

# Simulation of Starting and Windmilling

A tutorial by Joachim Kurzke



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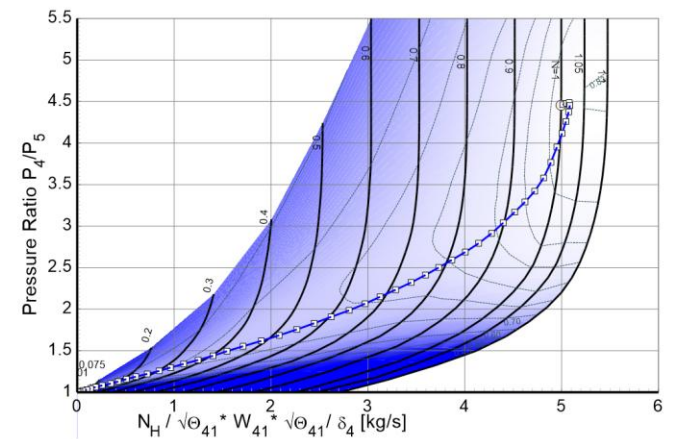
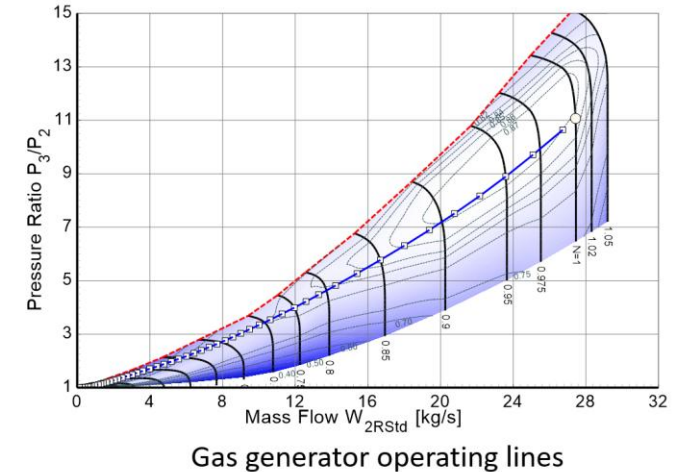


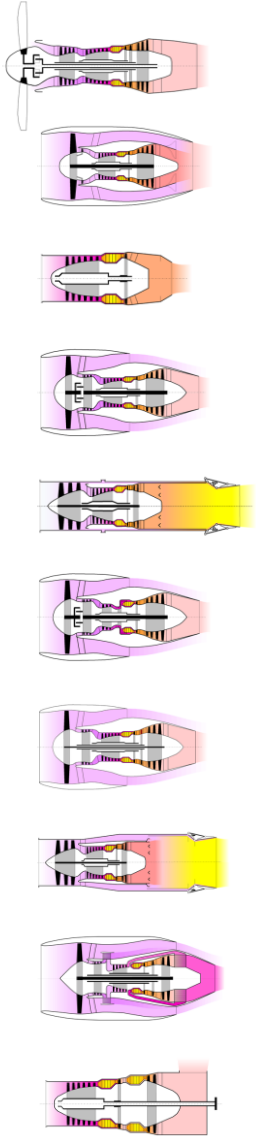
# Preface

It is a widespread view that reasonably accurate gas turbine performance calculations are only feasible between idle and max power.

However, the common performance programs can calculate the thermodynamic cycle for spool speeds as low as 1% of the design value provided physically sound compressor and turbine maps are available.

The 2<sup>nd</sup> edition of the book “Propulsion and Power” describes how to extend compressor and turbine maps from the idle region down to extremely low spool speeds. With such maps it is feasible to simulate engine starting and windmilling.





# Outline

This tutorial is intended to support readers of Chapter 17 in "Propulsion and Power" who want to carry out their own calculations on starting and windmilling.

The first part of the tutorial describes the steps involved in simulating the start-up of an aircraft engine. Although the specific algorithms in your favorite performance program may differ, the underlying principles remain the same as described here.

The presented example simulation is supplemented by sensitivity studies, which illustrate the importance of the various input data.

The second part of this tutorial is dedicated to the simulation of windmilling. It also covers relighting an engine from the windmilling state.



# Preparing an Engine Start Simulation

## 1 Create an engine model consisting of

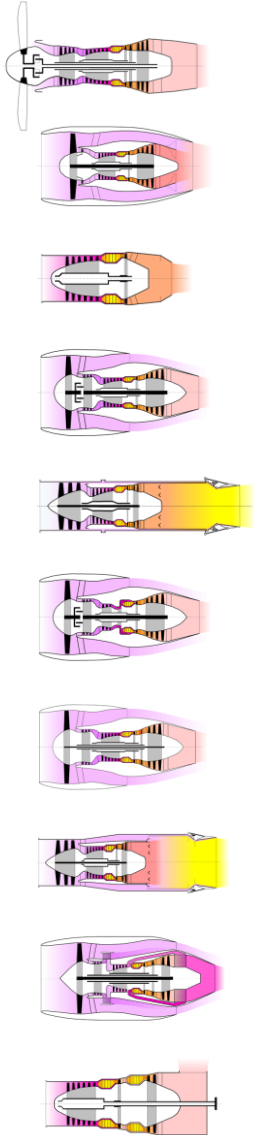
- The thermodynamic cycle design point
  - down to very low speed (10% ... 1%)
- Extended compressor and turbine maps
- Engine geometry description (yields a guess of rotor inertia)

## 2 Check the thermodynamic model

- Intake pressure ratio = 1 at zero spool speed?
- Customer Bleed = 0 kg/s at zero spool speed?
- There must not be losses which do not decrease with spool speed!

## 3 Search for a suitable point to initialize the simulation

- Zero spool speed is trivial, but no cycle calculation feasible
- A crank point at very low spool speed is required
  - run a 30% spool speed case with fuel
  - switch to crank mode
  - run an operating line in crank mode down to minimum simulatable spool speed



# Cycle Design Point

## Gas Generator Model of the CFM56-3

Station	W kg/s	T K	P kPa	WRstd kg/s
amb		288.15	101.325	
1	27.436	288.15	101.325	
2	27.436	288.15	101.325	27.436
3	27.436	609.27	1121.668	3.604
31	23.568	609.27	1121.668	
4	23.973	1228.40	1065.584	4.707
41	25.894	1185.80	1065.584	4.995
49	25.894	886.65	239.176	
5	27.540	870.78	239.176	20.282
6	27.540	870.78	239.176	
8	27.540	870.78	239.176	20.282
Bleed	0.302	609.27	1121.668	

P2/P1 = 1.0000	P4/P3 = 0.9500	P6/P5 = 1.0000	
Efficiencies:	isent	polytr	RNI P/P
Compressor	0.8677	0.9033	1.000 11.070
Burner	0.9995		
Turbine	0.8250	0.7960	2.003 4.455

Spool mech Eff	0.9900	Nom Spd	13498 rpm
----------------	--------	---------	-----------

hum [%]	war0	FHV	Fuel
0.0	0.00000	42.769	Generic

SL static, ISA

FN	=	17.08 kN
TSFC	=	23.7534 g/(kN*s)
FN/W2	=	622.66 m/s

Prop Eff	=	0.0000
eta core	=	0.3060

WF	=	0.40579 kg/s
s NOx	=	0.24987
XM8	=	1.0000
A8	=	0.0854 m²
P8/Pamb	=	2.3605
WBld/w2	=	0.01100
Ang8	=	1.00 deg
CD8	=	0.9980
wCln/w2	=	0.07000
wClr/w2	=	0.06000
Loading	=	100.00 %
e45 th	=	0.79974
far8	=	0.01495
PWX	=	0.00 kW

Cycle Design Point

Net Thrust

Thrust Specific Fuel Consumption

Specific Thrust

Propulsion Efficiency

Core Efficiency

Fuel Flow

NOx Severity Parameter

Nozzle Throat Mach No.

Geometric Nozzle Throat Area

Nozzle Pressure Ratio

Bleed Air Flow/Mass Flow w2

Nozzle Petal Angle

Nozzle Discharge Coefficient

Turbine Nozzle Guide Vane Cooling Air / w2

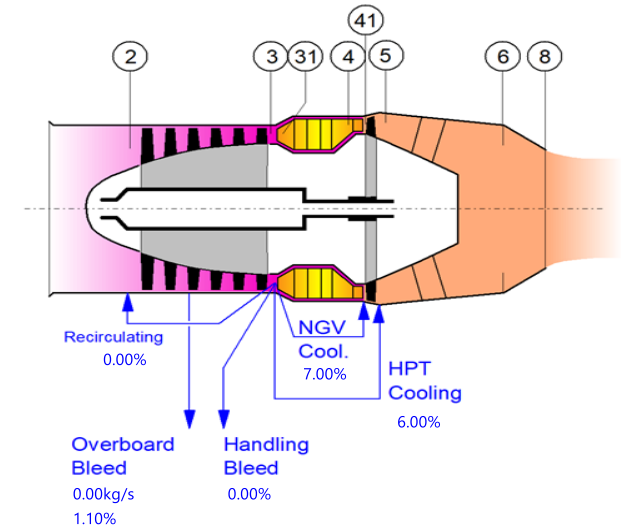
Turbine Rotor Cooling Air/ w2

Burner Loading in % of the Cycle Design Point Value

Thermodynamic Turbine Efficiency

Nozzle Throat Fuel-Air-Ratio

Power Offtake



Ref.

Foil# 12 in [https://kurzke-consulting.de/images/Tutorials/TurboExpo2025\\_Tutorial.pdf](https://kurzke-consulting.de/images/Tutorials/TurboExpo2025_Tutorial.pdf)



## Extended Maps

Download from

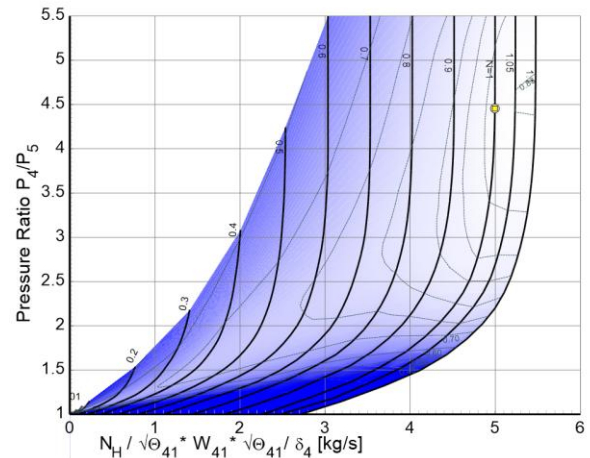
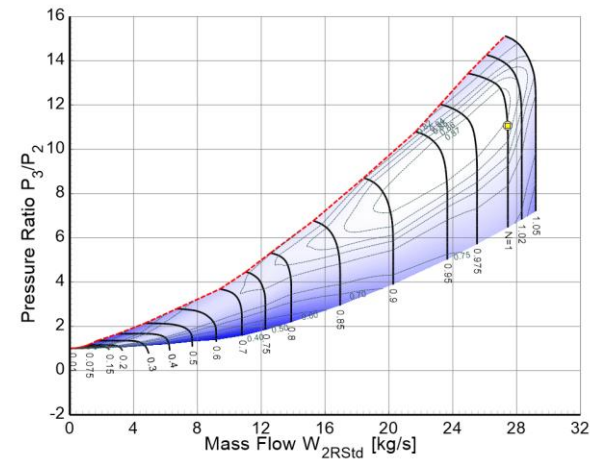
<https://kurzke-consulting.de/my-products/compressor-maps/isabe-2007-1184.html>

ISABE2007\_1184\_Fig29\_ExtndGasTurb.txt

Save as

ISABE2007\_1184\_Fig29\_ExtndGasTurb.Map

- Set the map scaling point to  $N=1, \beta=0.5$



Download from

<https://kurzke-consulting.de/my-products/turbine-maps/nasa-tm-83665.html>

NASATM83655\_ExtndTurbine.txt

Save as

NASATM83655\_ExtndTurbine.Map

- Set the map scaling point to  $N=1$ ,  $\beta=0.5$



# Steady State Operation

## 30% Spool Speed

Station	W kg/s	T K	P kPa	WRstd kg/s
amb		288.15	101.325	
1	2.313	288.15	101.325	
2	2.313	288.15	101.325	2.313
3	2.313	337.54	136.838	1.853
31	1.987	337.54	136.838	
4	2.027	1087.70	135.028	2.955
41	2.189	1037.21	135.028	3.116
49	2.189	992.25	102.390	
5	2.327	956.36	102.390	4.196
6	2.327	956.36	102.390	
8	2.327	956.36	102.390	4.196
Bleed	0.025	337.54	136.838	
-----				
P2/P1 = 1.0000	P4/P3 = 0.9868	P6/P5 = 1.0000		
Efficiencies:	isent	polytr	RNI	P/P
Compressor	0.5217	0.5415	1.000	1.350
Burner	0.9826			0.987
Turbine	0.6663	0.6588	0.296	1.319
-----				
Spool mech Eff	0.9900	Speed	30.00 %	
-----				
hum [%]	war0	FHV	Fuel	
0.0	0.00000	42.769	Generic	

SL static, ISA , Rel GG Speed=0.300

FN	=	0.18 kN	Net Thrust
TSFC	=	227.3385 g/(kN*s)	Thrust Specific Fuel Consumption
FN/W2	=	76.22 m/s	Specific Thrust
Prop Eff	=	0.0000	Propulsion Efficiency
eta core	=	0.0039	Core Efficiency
P5/P2	=	1.0105 EPR	Engine Pressure Ratio
WF	=	0.04007 kg/s	Fuel Flow
s NOx	=	0.02662	NOx Severity Parameter
XM8	=	0.1256	Nozzle Throat Mach No.
A8	=	0.0881 m²	Geometric Nozzle Throat Area
P8/Pamb	=	1.0105	Nozzle Pressure Ratio
WBld/w2	=	0.01100	Bleed Air Flow/Mass Flow w2
Ang8	=	16.00 deg	Nozzle Petal Angle
CD8	=	0.9427	Nozzle Discharge Coefficient
WClN/w2	=	0.07000	Turbine Nozzle Guide Vane Cooling Air / w2
WClR/w2	=	0.06000	Turbine Rotor Cooling Air/ w2
Loading	=	919.88 %	Burner Loading in % of the Cycle Design Point value
e45 th	=	0.65477	Thermodynamic Turbine Efficiency
far8	=	0.01752	Nozzle Throat Fuel-Air-Ratio
PWX	=	0.00 kw	Power Offtake

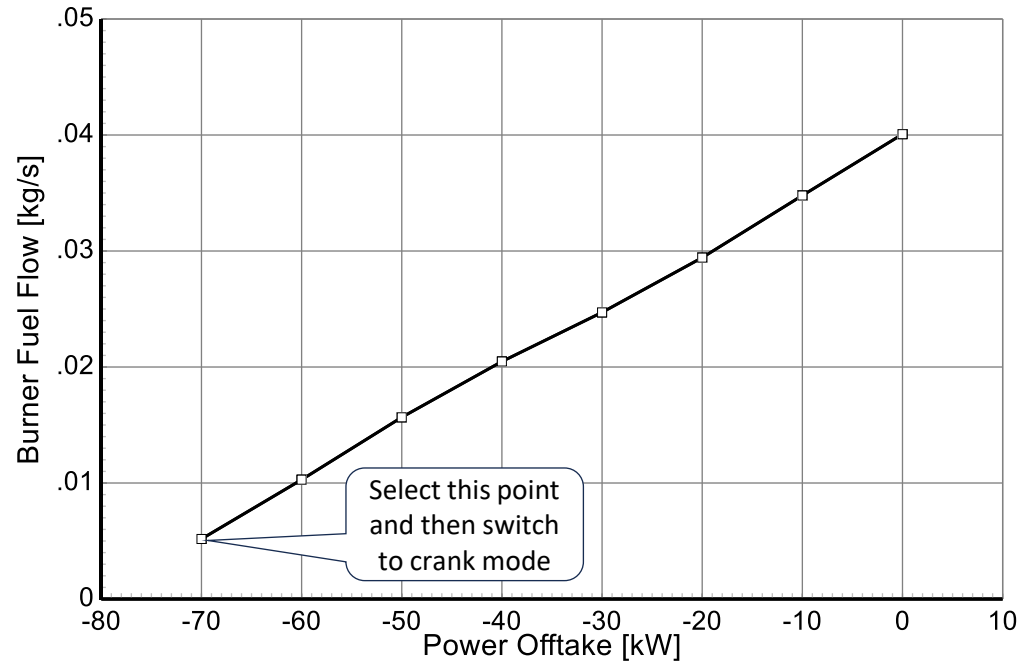
### Standard off-design iteration setup for a given speed

Variable	Error
Beta in HPC Map	Turbine Flow
T4	Turbine Pressure Ratio
Beta in HPT Map	Nozzle Pressure Ratio



# Parametric Study With Negative PWX

Power Offtake = 0 ... -70 [kW]



One might encounter convergence problems when switching from a normal steady-state performance point with a given spool speed (fuel flow > 0) to a crank point (fuel flow = 0).

In such a case, run a parametric study with decreasing power offtake. Select a point with low fuel flow from this study, and then switch to crank mode.





# Steady State Operation

## 30% Spool Speed, No Fuel - Crank

Station	W kg/s	T K	P kPa	WRstd kg/s
amb		288.15	101.325	
1	3.867	288.15	101.325	
2	3.867	288.15	101.325	3.867
3	3.867	320.63	136.133	3.036
31	3.322	320.63	136.133	
4	3.322	320.63	131.302	2.704
41	3.593	320.63	131.302	2.925
49	3.593	307.68	101.686	
5	3.825	308.47	101.686	3.943
6	3.825	308.47	101.686	
8	3.825	308.47	101.686	3.943
Bleed	0.043	320.63	136.133	
-----				
P2/P1 = 1.0000	P4/P3 = 0.9645	P6/P5 = 1.0000		
Efficiencies:	isent	polytr	RNI	P/P
Compressor	0.7797	0.7887	1.000	1.344
Burner	0.0000			0.965
Turbine	0.5752	0.5662	1.141	1.291
-----				
Spool mech Eff	0.9900	Speed	30.00 %	
-----				
hum [%]	war0	FHV	Fuel	

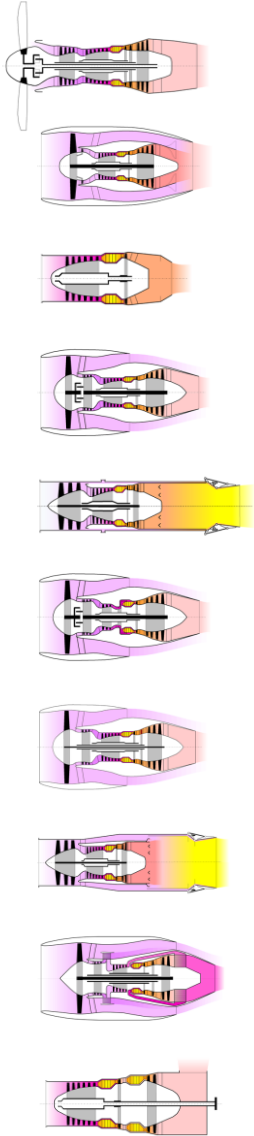
SL static, ISA , Re1 GG Speed=0.300, **crank**

FN	=	0.10 kN	Net Thrust
TSFC	=	0.0000 g/(kN*s)	Thrust Specific Fuel Consumption
FN/W2	=	24.84 m/s	Specific Thrust
Prop Eff	=	0.0000	Propulsion Efficiency
eta core	=	0.0000	Core Efficiency
P5/P2	=	1.0036 EPR	Engine Pressure Ratio
<b>WF</b>	<b>=</b>	<b>0.00000 kg/s</b>	<b>Fuel Flow</b>
s NOx	=	0.00000	NOx Severity Parameter
XM8	=	0.0714	Nozzle Throat Mach No.
A8	=	0.0881 m²	Geometric Nozzle Throat Area
P8/Pamb	=	1.0036	Nozzle Pressure Ratio
WBld/W2	=	0.01100	Bleed Air Flow/Mass Flow w2
Ang8	=	16.00 deg	Nozzle Petal Angle
CD8	=	0.9783	Nozzle Discharge Coefficient
WC1N/W2	=	0.07000	Turbine Nozzle Guide Vane Cooling Air / w2
WC1R/W2	=	0.06000	Turbine Rotor Cooling Air/ w2
Loading	=	919.88 %	Burner Loading in % of the Cycle Design Point value
e45 th	=	0.00000	Thermodynamic Turbine Efficiency
far8	=	0.00000	Nozzle Throat Fuel-Air-Ratio
<b>PWX</b>	<b>=</b>	<b>-79.99 kw</b>	<b>Power Offtake</b>

Crank iteration setup, spool speed is given

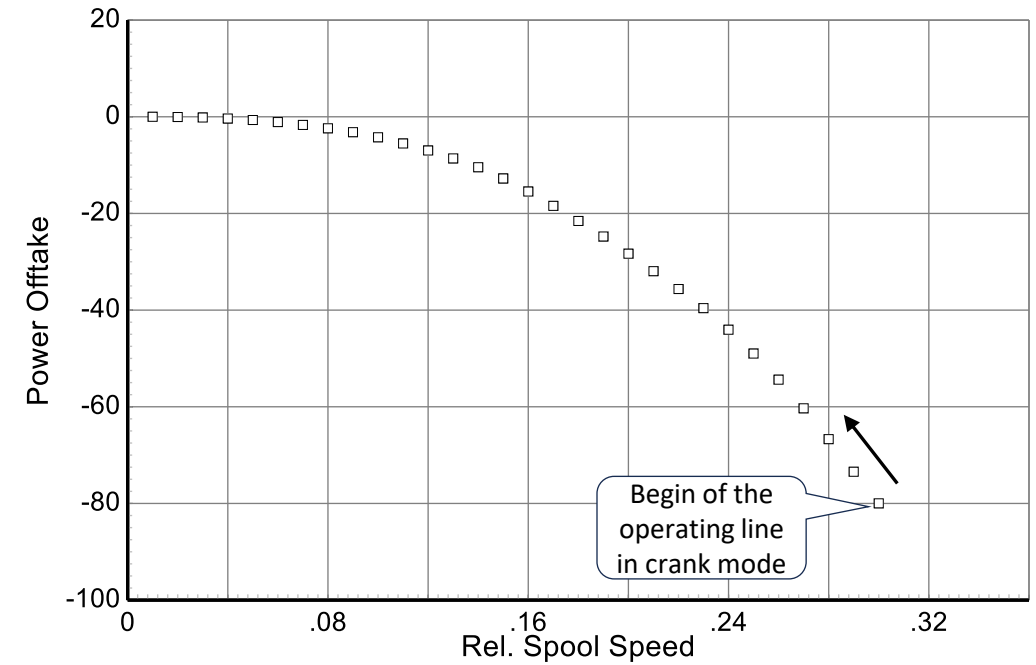
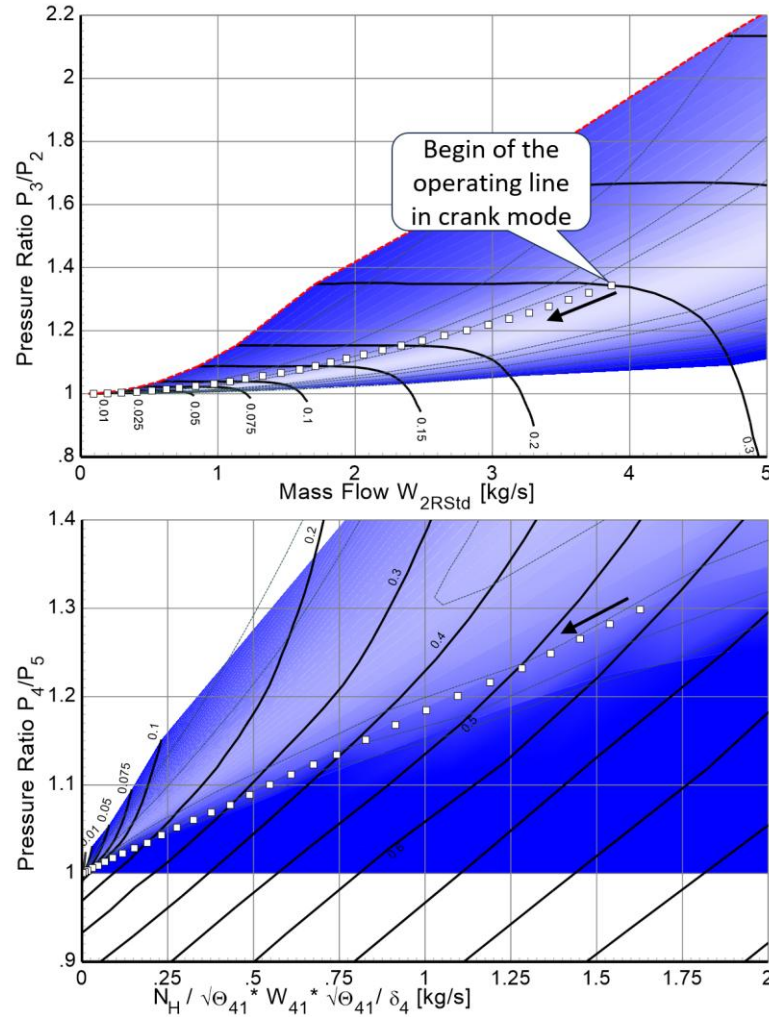
Variable	Error
Beta in HPC Map	Turbine Flow
PWX	Turbine Pressure Ratio
Beta in HPT Map	Nozzle Pressure Ratio





# Operating Points in Crank Mode

## From 30% Spool Speed Down to 1%



# Crank - 1% Spool Speed

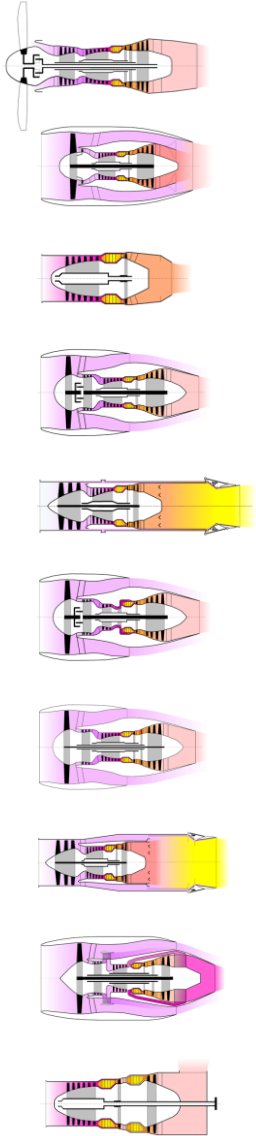
## Begin of the Engine Start Simulation

Station	W kg/s	T K	P kPa	WRstd kg/s
amb		288.15	101.325	
1	0.094	288.15	101.325	
2	0.094	288.15	101.325	0.094
3	0.094	288.20	101.364	0.094
31	0.081	288.20	101.364	
4	0.081	288.20	101.361	0.081
41	0.087	288.20	101.361	0.087
49	0.087	288.25	101.326	
5	0.093	288.24	101.326	0.093
6	0.093	288.24	101.326	
8	0.093	288.24	101.326	0.093
Bleed	0.001	288.20	101.364	
-----				
P2/P1 = 1.0000	P4/P3 = 1.0000	P6/P5 = 1.0000		
Efficiencies:	isent	polytr	RNI	P/P
Compressor	0.5964	0.5953	1.000	1.000
Burner	0.0000			1.000
Turbine	-1.5059	-1.5061	1.000	1.000
-----				
Spool mech Eff	0.9900	Speed	1.00 %	
-----				
hum [%]	war0	FHV	Fuel	
0.0	0.00000	42.769	Generic	

SL static, ISA , Rel GG Speed=0.010, **crank**

FN	=	9.1E-5 kN	Net Thrust
TSFC	=	0.0000 g/(kN*s)	Thrust Specific Fuel Consumption
FN/W2	=	0.97 m/s	Specific Thrust
Prop Eff	=	0.0000	Propulsion Efficiency
eta core	=	0.0000	Core Efficiency
P5/P2	=	1.0000 EPR	Engine Pressure Ratio
<b>WF</b>	<b>=</b>	<b>0.00000 kg/s</b>	<b>Fuel Flow</b>
s NOx	=	0.00000	NOx Severity Parameter
XM8	=	0.0029	Nozzle Throat Mach No.
A8	=	0.0881 m²	Geometric Nozzle Throat Area
P8/Pamb	=	1.0000	Nozzle Pressure Ratio
WBld/w2	=	0.01100	Bleed Air Flow/Mass Flow w2
Ang8	=	16.00 deg	Nozzle Petal Angle
CD8	=	0.9984	Nozzle Discharge Coefficient
WClN/w2	=	0.07000	Turbine Nozzle Guide Vane Cooling Air / w2
WClR/w2	=	0.06000	Turbine Rotor Cooling Air/ w2
Loading	=	99.99 %	Burner Loading in % of the Cycle Design Point Value
e45 th	=	0.00000	Thermodynamic Turbine Efficiency
far8	=	0.00000	Nozzle Throat Fuel-Air-Ratio
<b>PWX</b>	<b>=</b>	<b>-0.01 kW</b>	<b>Power Offtake</b>

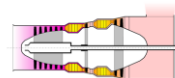
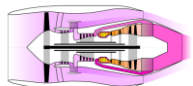
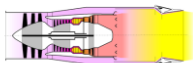
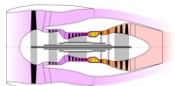
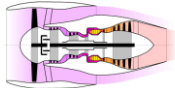
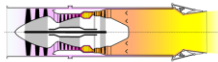
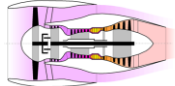
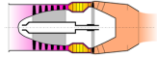
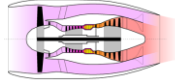
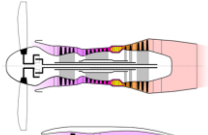




# Engine Start Simulation

A Special Case of a Transient Manouver





# How it Works

Engine starting is a transient event that begins with a steady-state crank point at very low spool speed

## *Before light-up:*

The starter accelerates the rotor until light-up speed is reached

- The starter is the sole source of energy during that first phase
- Starter power can be described with two numbers: torque and spool speed
- A power limit can exist

## *After light-up:*

Both the starter and the combustion of fuel add energy to the cycle

- The amount of fuel is determined by a controller with the following limiters
  - Accel  $\dot{N}/\delta$  Limit
  - Minimum fuel-air-ratio  $far_{min}$
  - Maximum fuel-air-ratio  $far_{max}$
  - Idle spool speed
- Fuel flow is not an input, it is a result
  - Burner efficiency varies with burner loading
  - The dubious accuracy of the burner efficiency model has only a minor impact on the overall result.





# Transient Performance Simulation

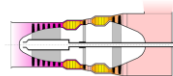
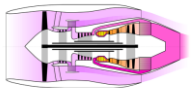
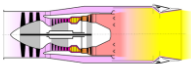
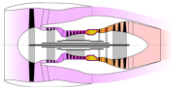
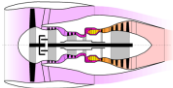
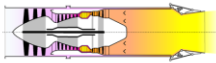
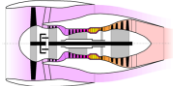
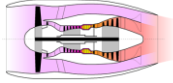
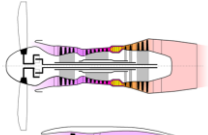
## Input Data for a Reference Starting Process

Rotor Inertia	kg m <sup>2</sup>	5.69
Rel N for PLA 0% (Idle)		0.6
Proportional Control Constant		0.04
Integral Control Constant		0
Differential Control Constant		0.04
Gain Modifier		1
Minimum Fuel-Air-Ratio		0.003
Maximum Fuel-Air-Ratio		0.026
Decel (dN/dt)/(P2/Pstd) Limit		0.1
Accel (dN/dt)/(P2/Pstd) Limit		0.033
Rel. Burner Light-up Speed		0.18
Rel. Starter Cut-off Speed		0.3
Max Starter Torque	N m	150
Starter Torque Slope		-0.2
Max Starter Power	kW	40

Transient iteration setup, before light-up	
Variable	Error
<i>Beta in HPC Map</i>	<i>Turbine Flow</i>
<i>PWX</i>	<i>Turbine Pressure Ratio</i>
<i>Beta in HPT Map</i>	<i>Nozzle Pressure Ratio</i>

Transient iteration setup, after light-up	
Variable	Error
<i>Acceleration Rate</i>	<i>Control System</i>
<i>Fuel-Air-Ratio</i>	<i>Turbine Flow</i>
<i>Beta in Compressor Map</i>	<i>Turbine Pressure Ratio</i>
<i>Beta in Turbine Map</i>	<i>Nozzle Pressure Ratio</i>





# Transient Performance Simulation After Light-up

## Proportional-Integral-Differential Control

The proportional term of the speed control loop modulates the fuel flow according to

$$\Delta W_{f,P} = C_P (N_{Demand} - N)$$

while the integral term is calculated as

$$\Delta W_{f,I} = C_I \int (N_{Demand} - N) dt$$

Finally, the differential term is

$$\Delta W_{f,D} = C_D \frac{d(N_{Demand} - N)}{dt}$$

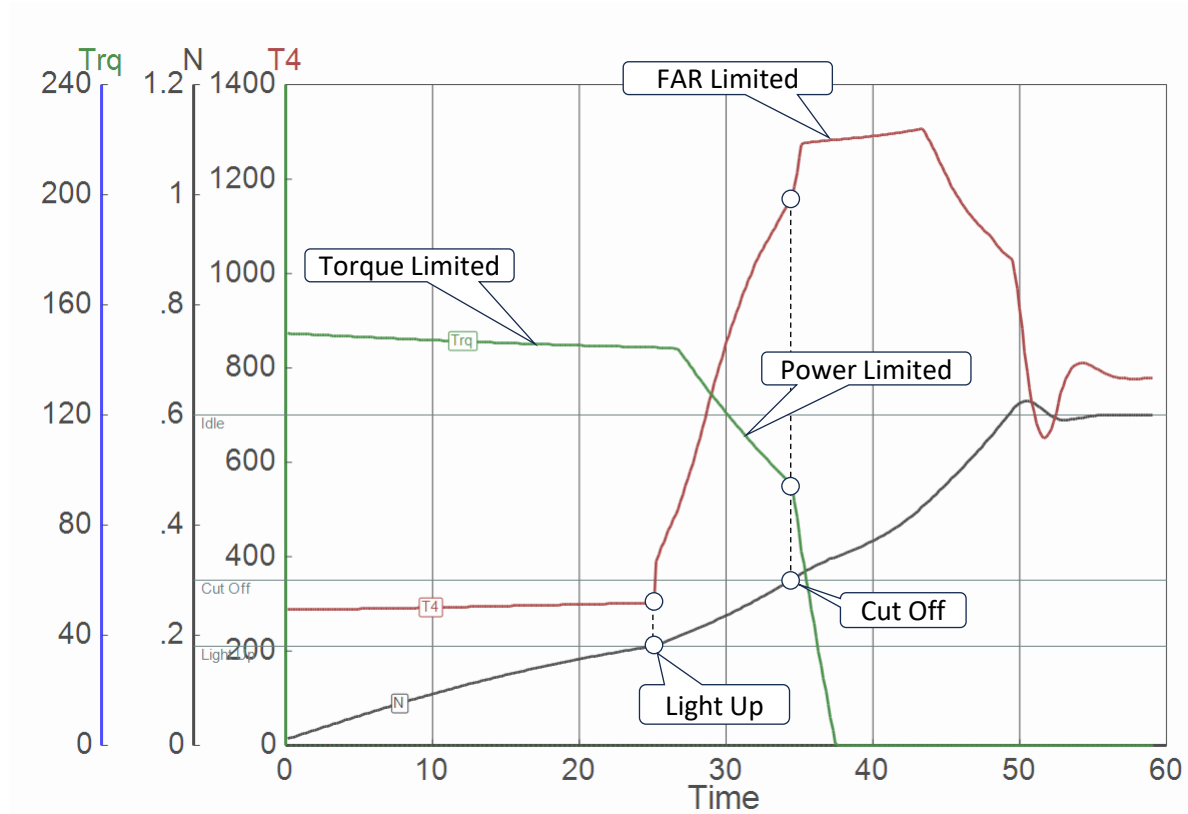
In a real engine, fuel flow is in principle controlled as described by these equations. In a simulation, however, it is better to control fuel flow indirectly via fuel-air-ratio because this is much less dependent on ambient conditions. Absolute fuel flow varies with engine inlet total pressure and temperature as well as spool speed, but the fuel-air-ratio is pretty much constant.

The operation of an engine model may be governed by setting selected combinations of the constants  $C_P$  (*Proportional Control Constant*),  $C_I$  (*Integral Control Constant*) and  $C_D$  (*Differential Control Constant*).

Note that the values for the control constants depend on the simulation algorithm: When fuel flow is the iteration variable, the control constants need to be adapted as engine inlet total pressure and temperature change because fuel mass flow is not a non-dimensional quantity. When fuel-air-ratio is the iteration variable, however, the control constants remain valid for a wide range of  $P_2$  and  $T_2$ .



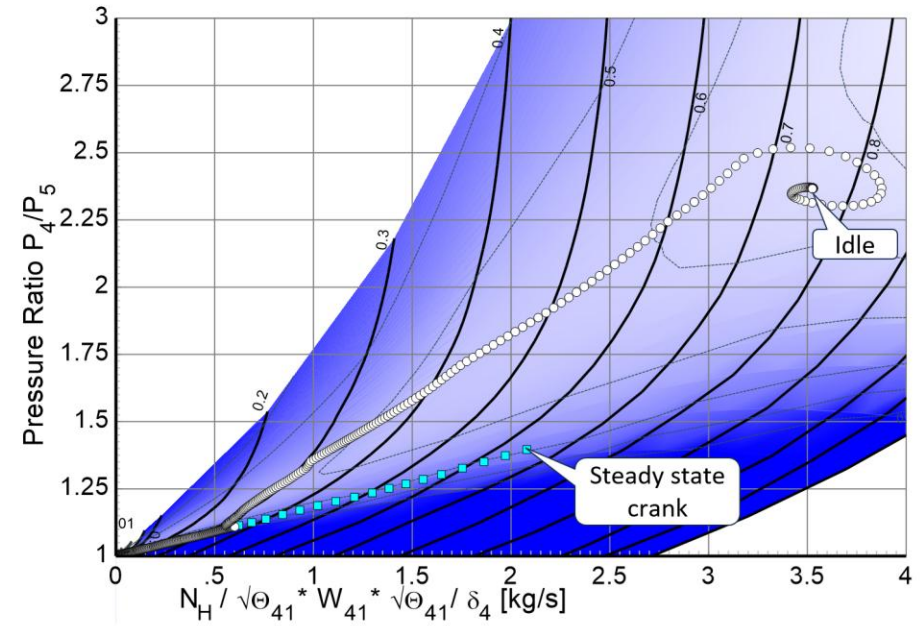
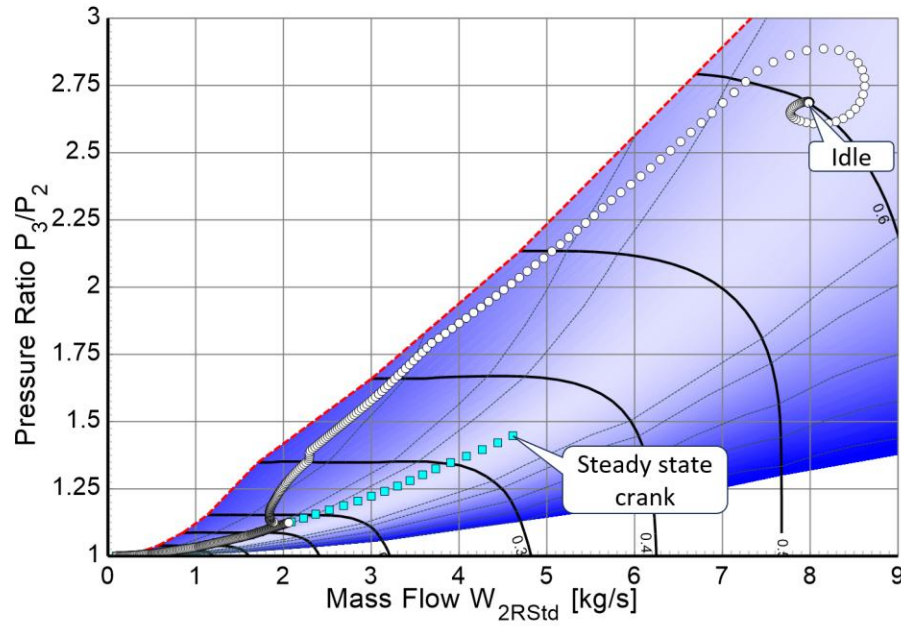
# Reference Starting Process

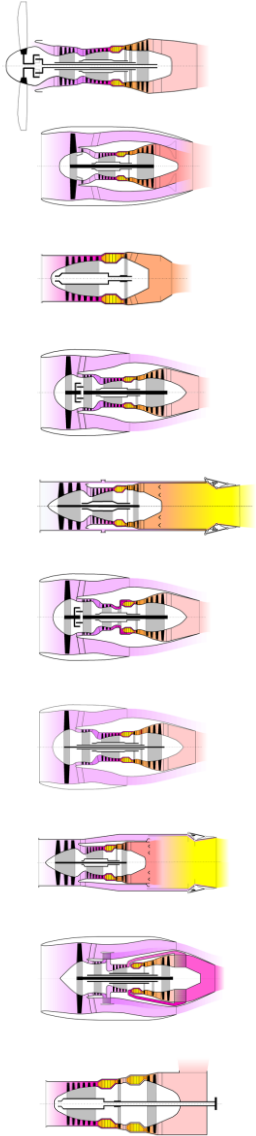


Timestep size = 0.165 sec



# Starting in the Component Maps





# Sensitivity Studies

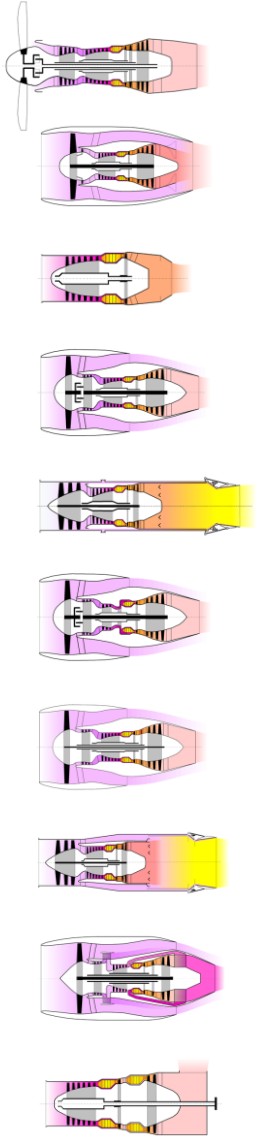
We will modify one input quantity at a time and compare the engine start history with our reference case.

Additionally, we demonstrate the influence of the combustion efficiency assumption on the result.

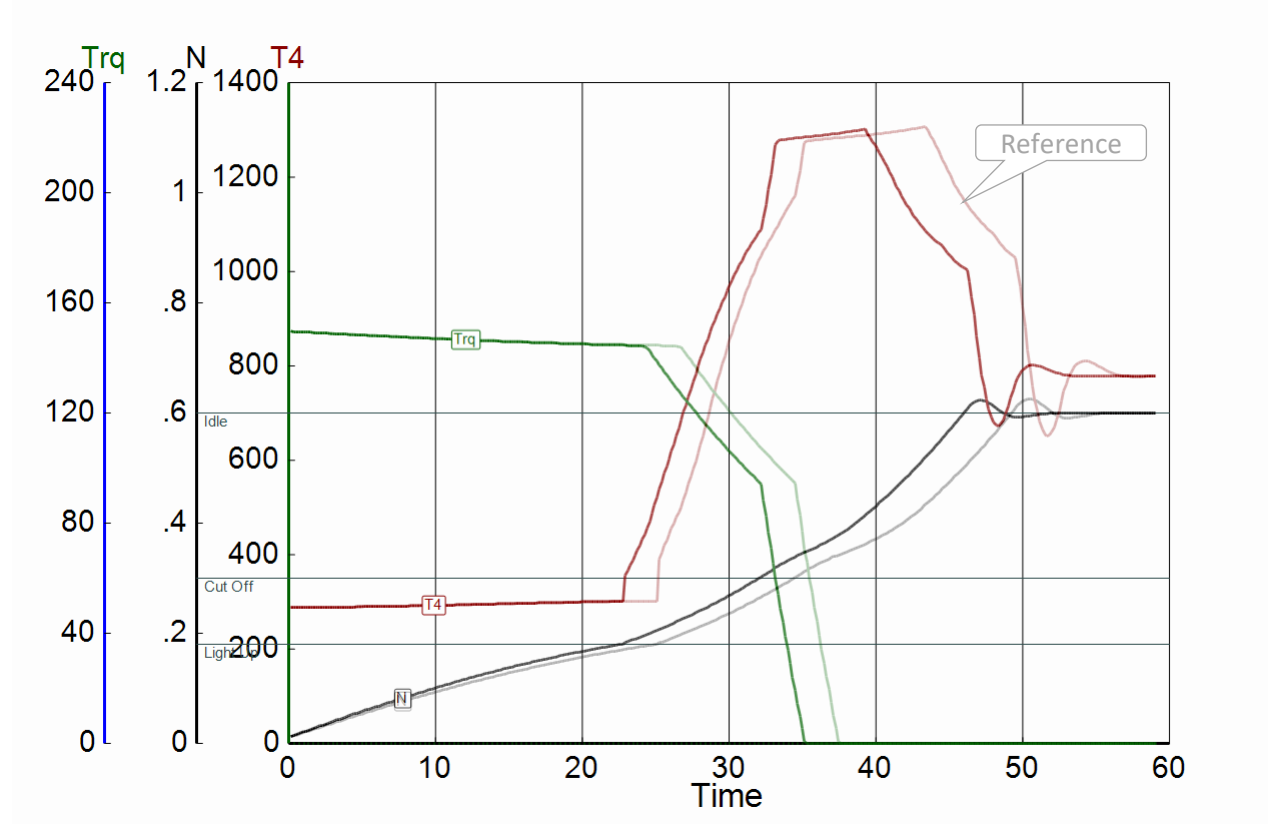
Compressor maps that extend down to 1% spool speed are rare. We also show how the simulation results are affected when the simulation begins with 10% instead of 1% spool speed.

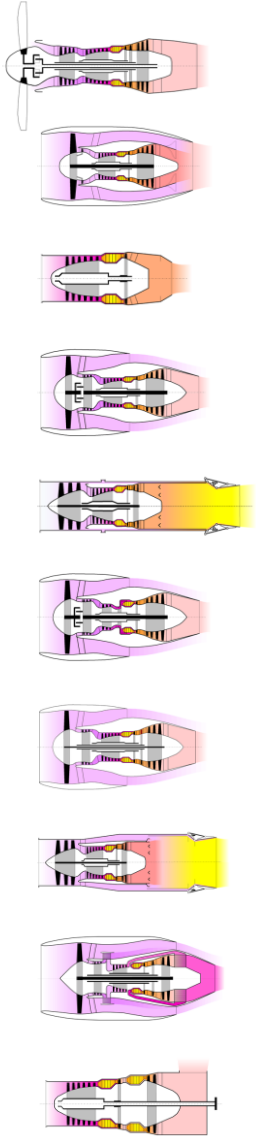




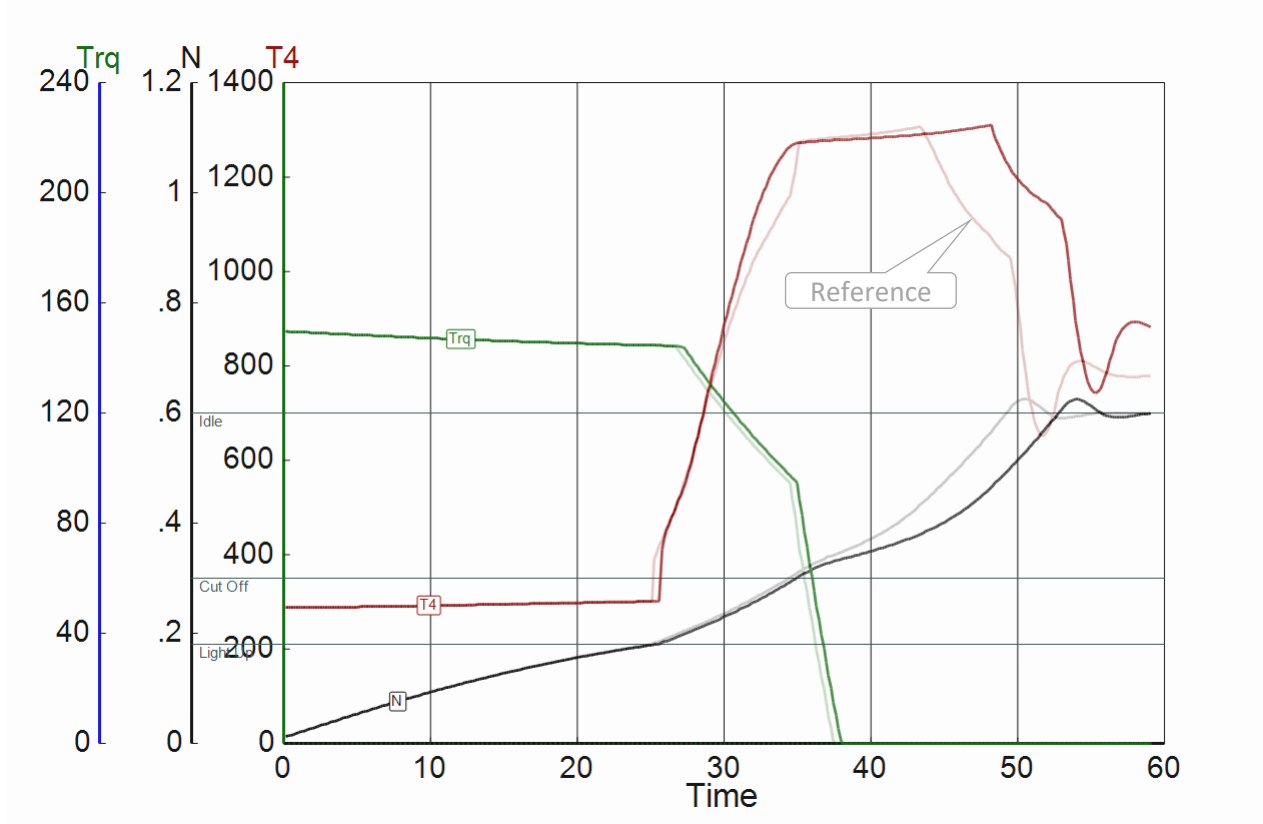


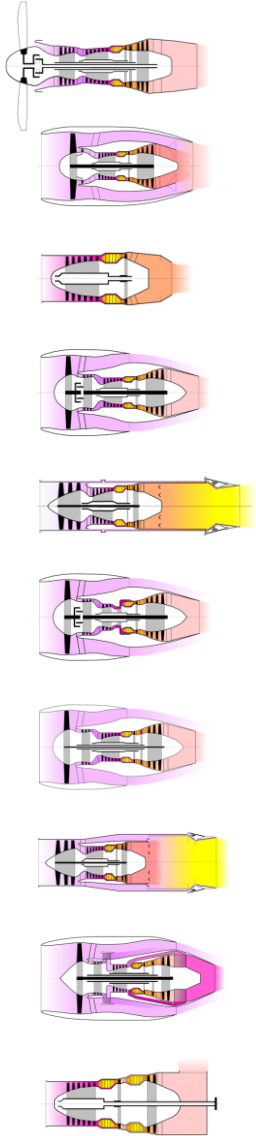
# Spool Inertia – 10%



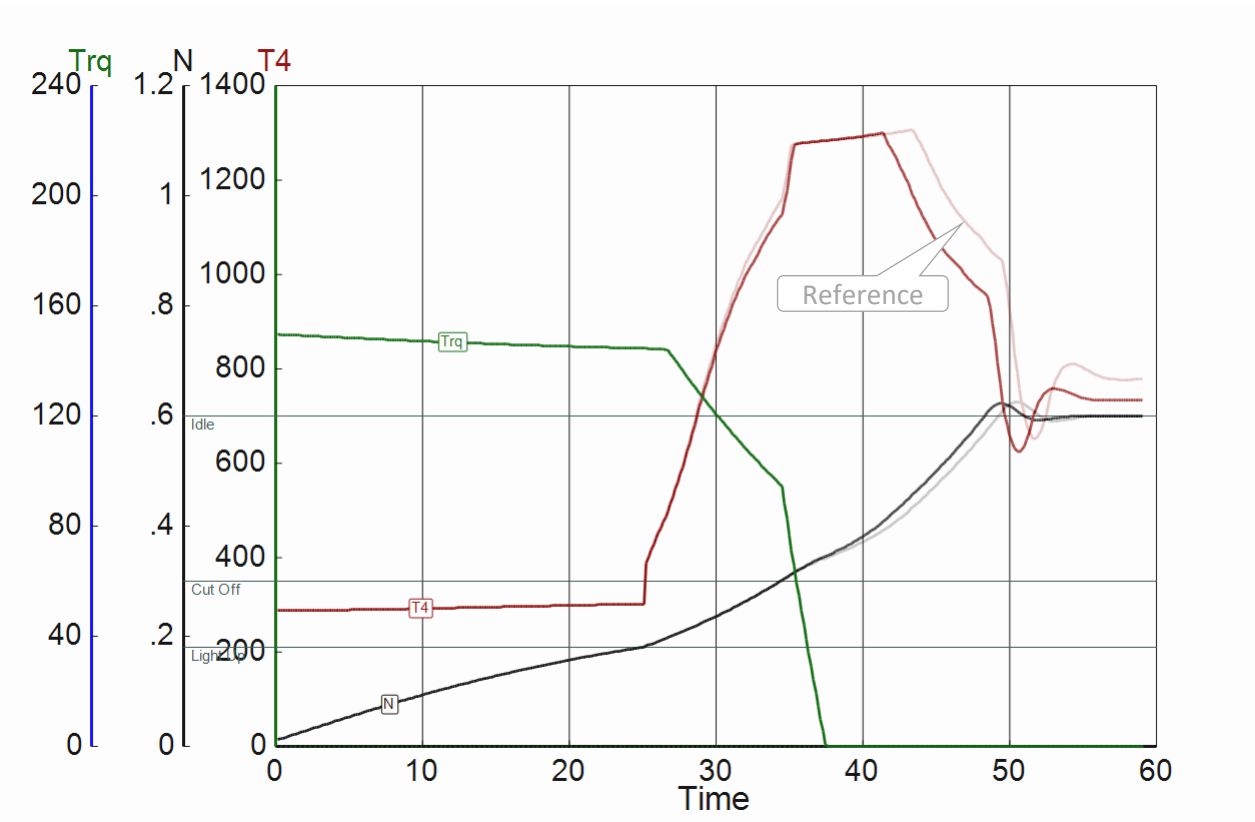


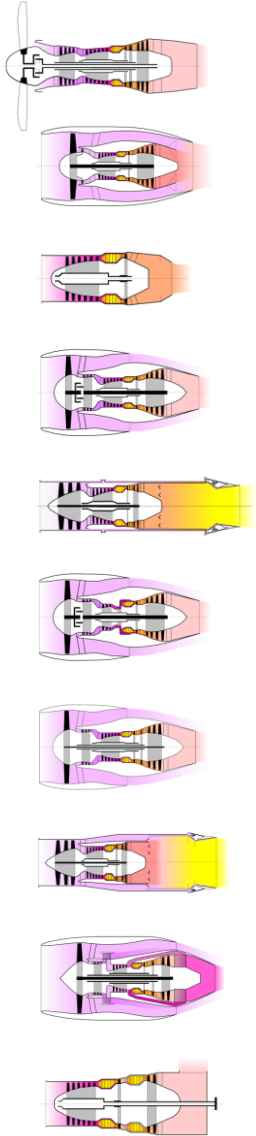
# 10% Handling Bleed





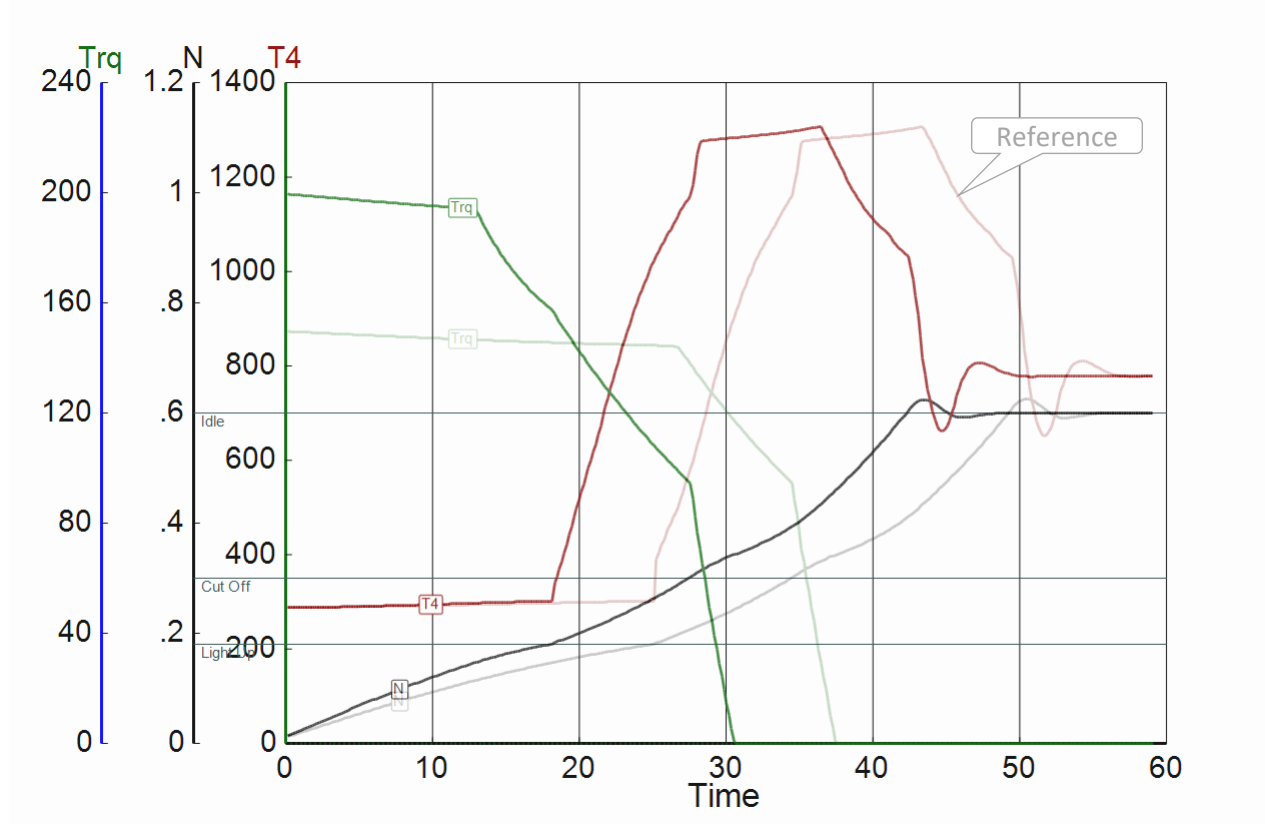
# Nozzle Area + 50%

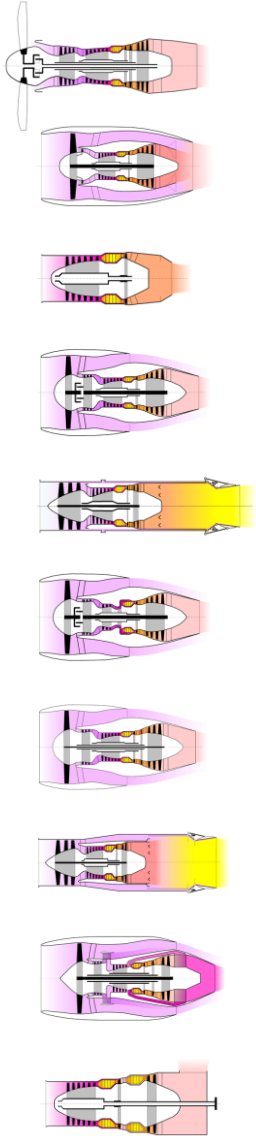




# Starter Torque

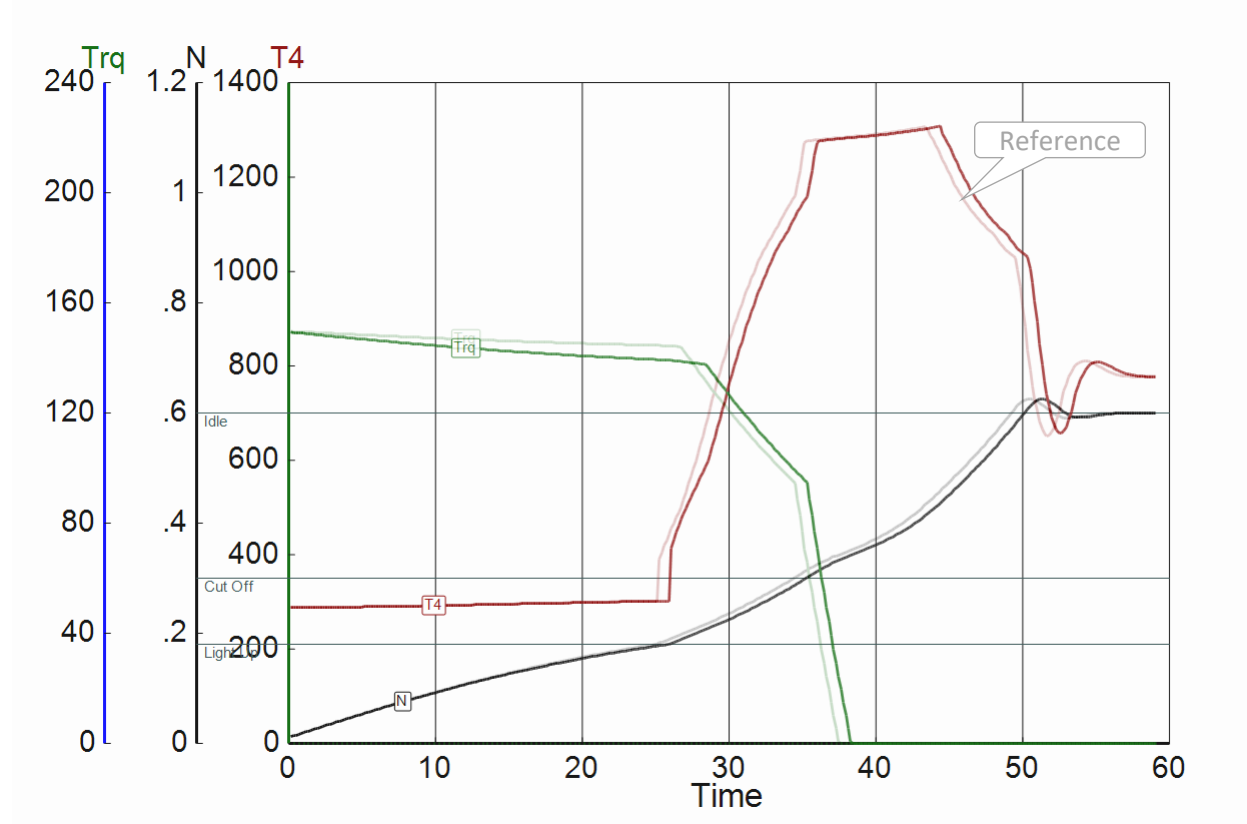
## From 150 to 200Nm



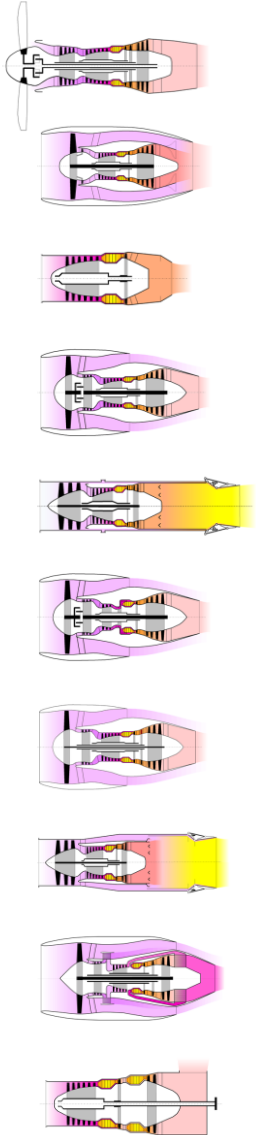


# Starter Torque Slope

## From -0.2 to -0.4

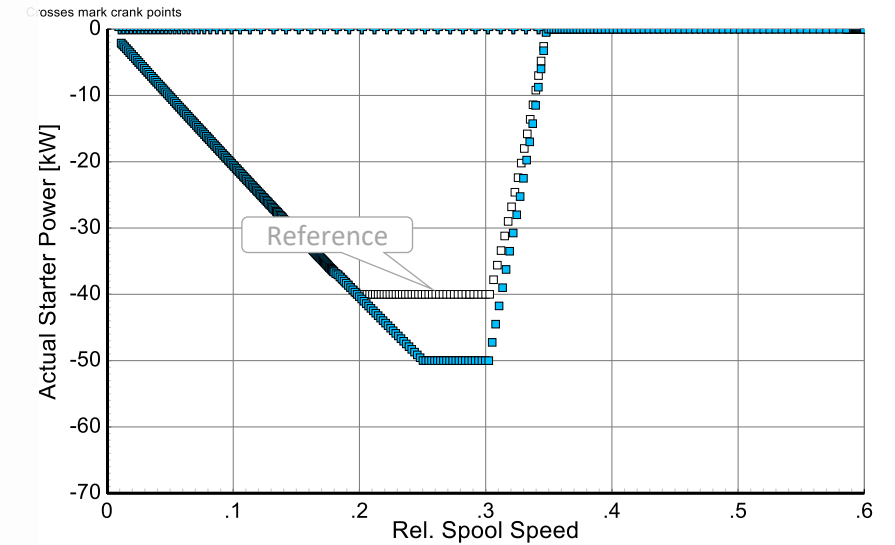
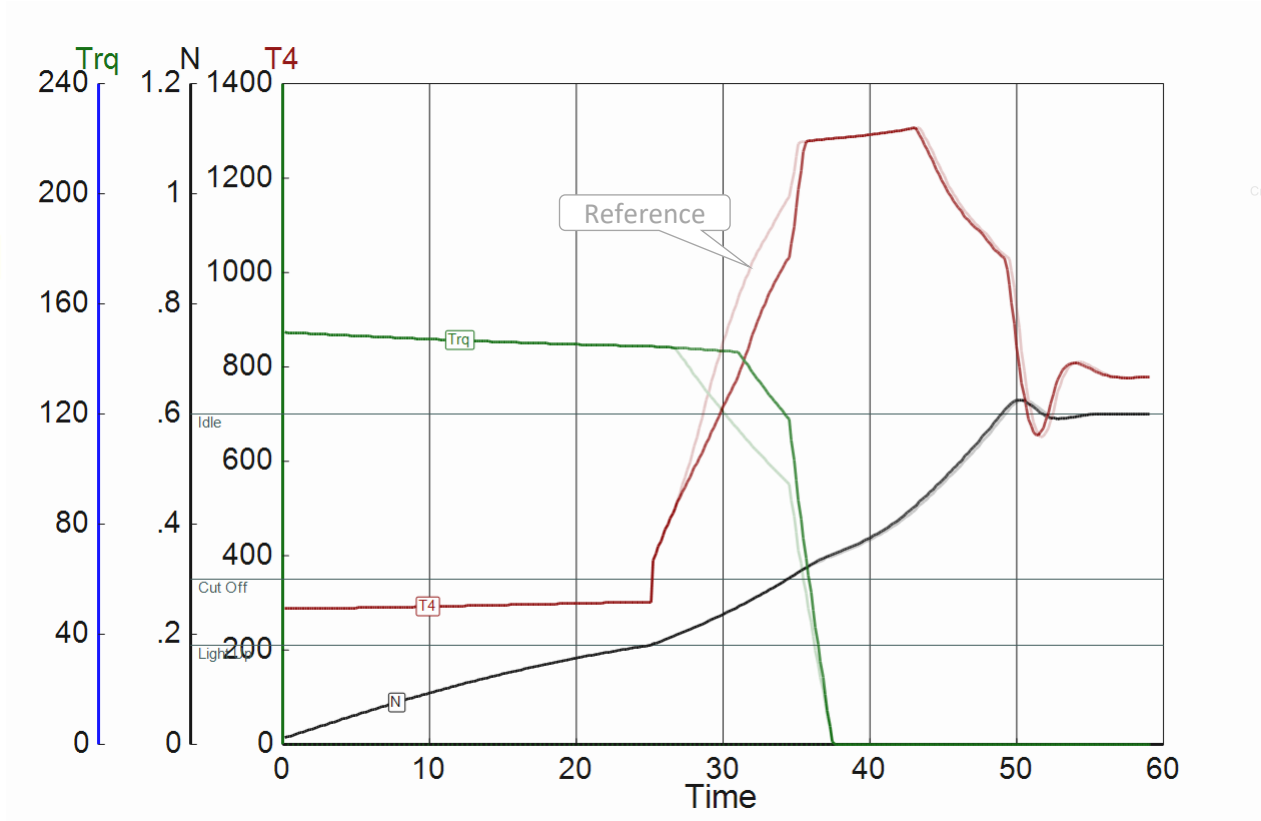


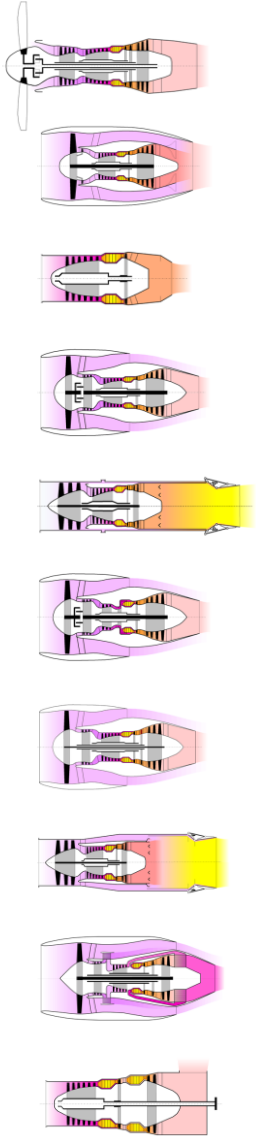




# Max Starter Power

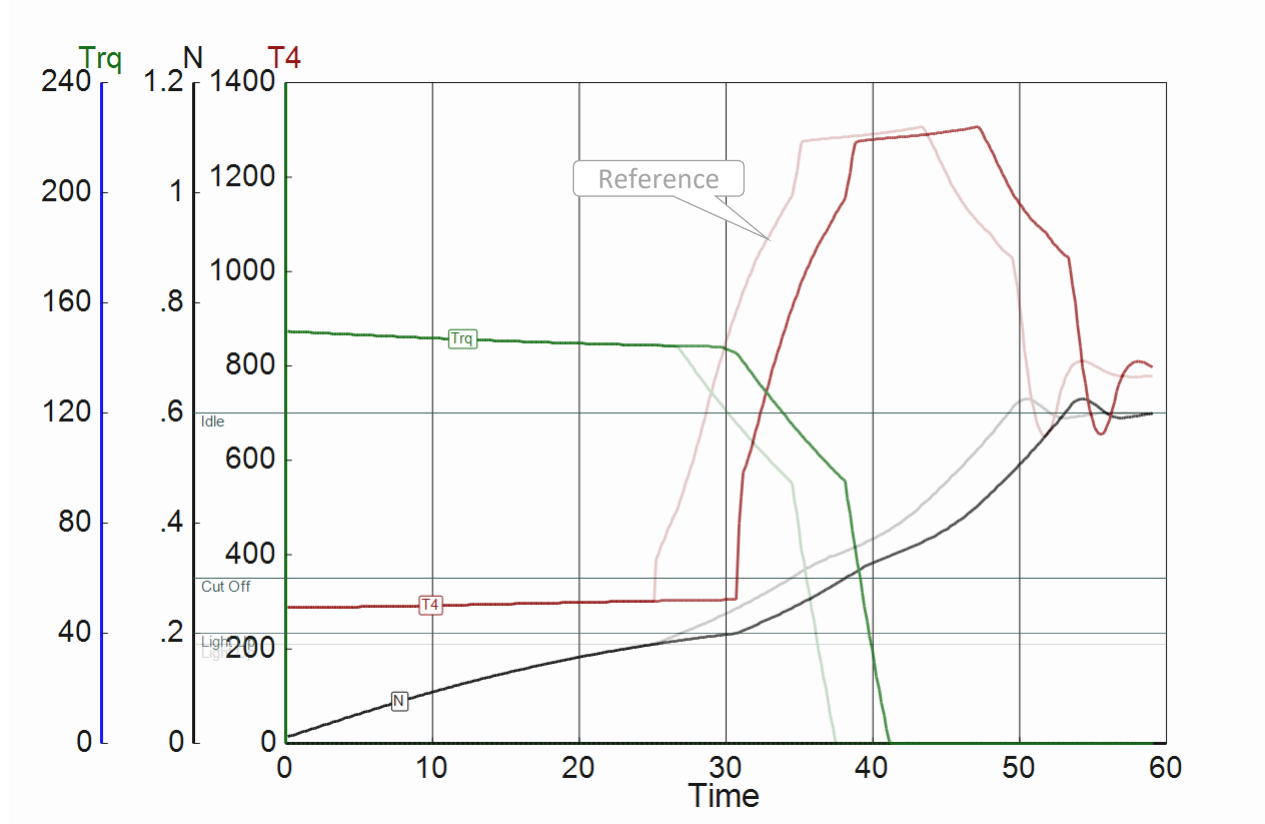
## From 40 to 50kW

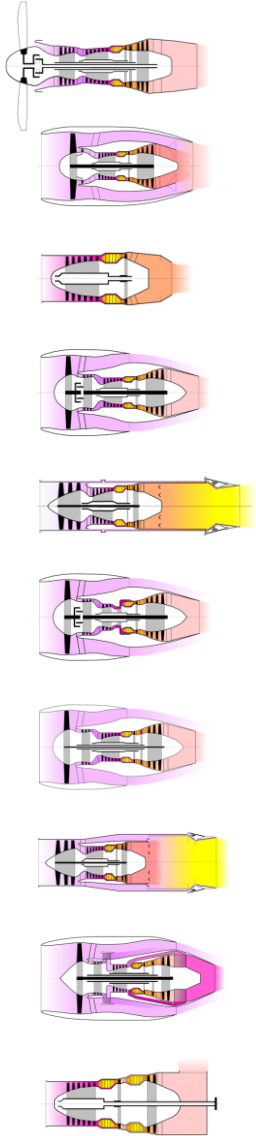




# Light-up Speed

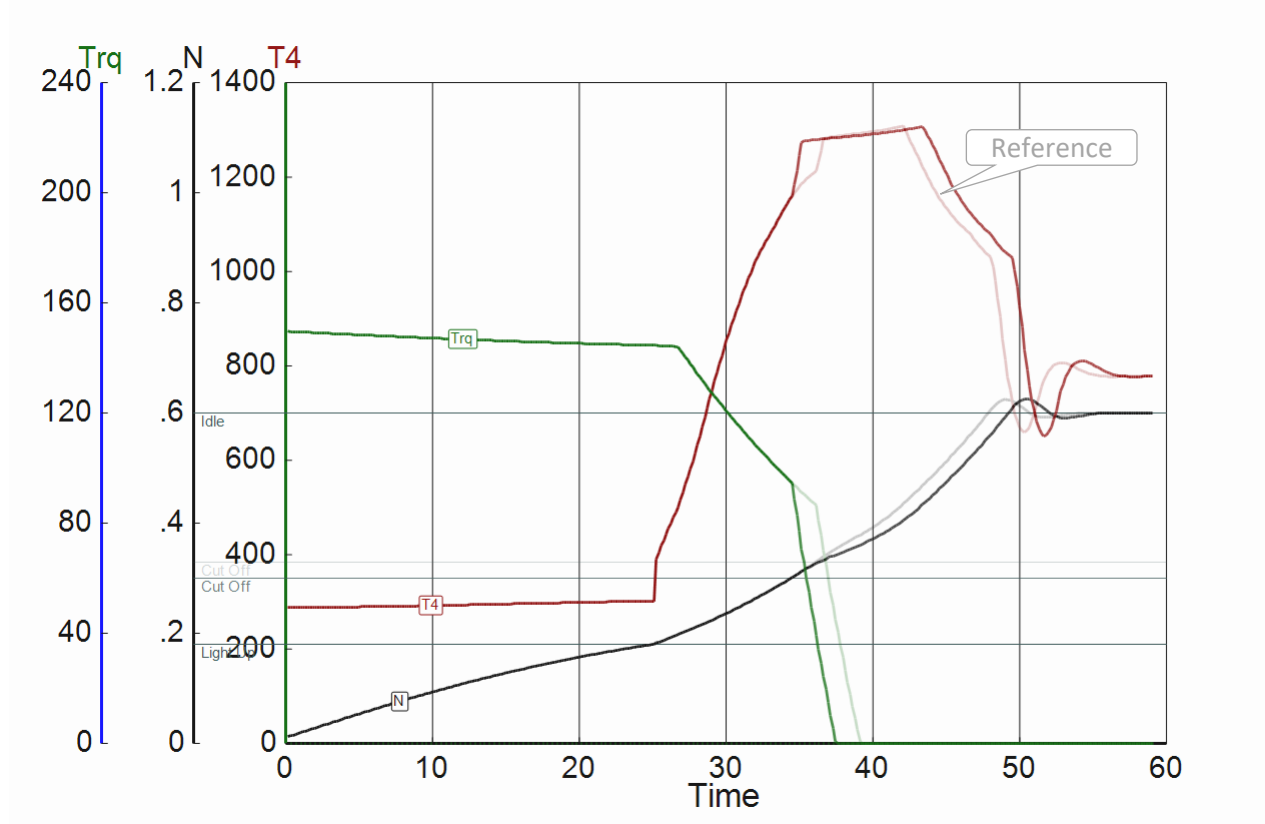
## From 0.18 to 0.2





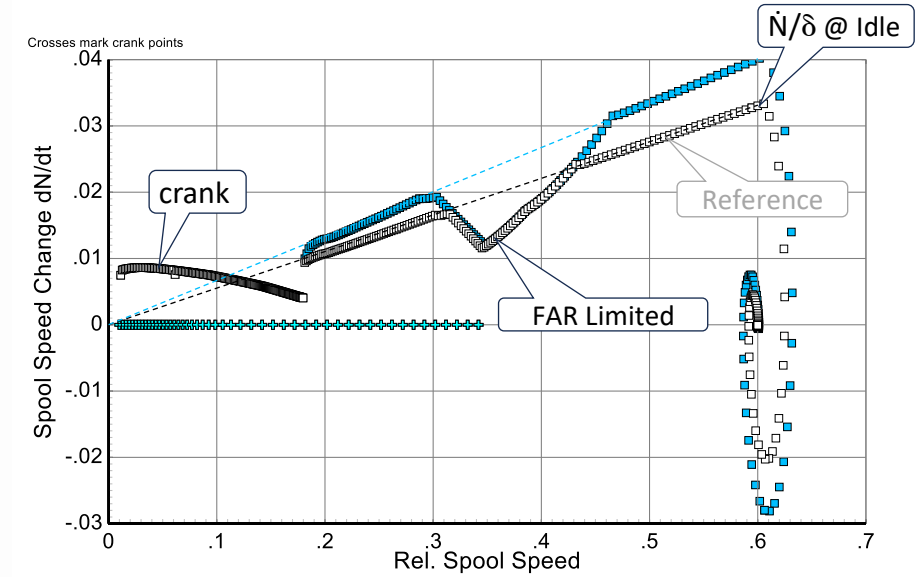
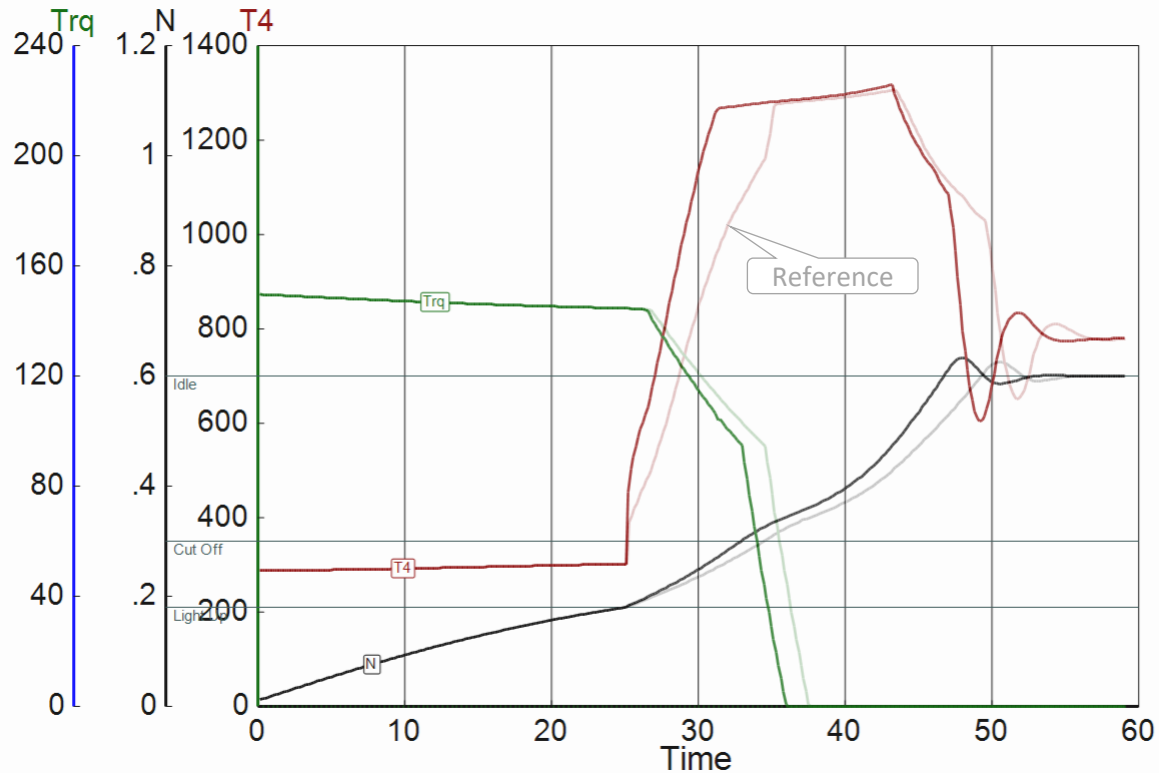
# Starter Cut-off Speed

## From 0.3 to 0.33



# Acceleration Rate Limiter $\dot{N}/\delta$ @ Idle

## From 0.033 to 0.04



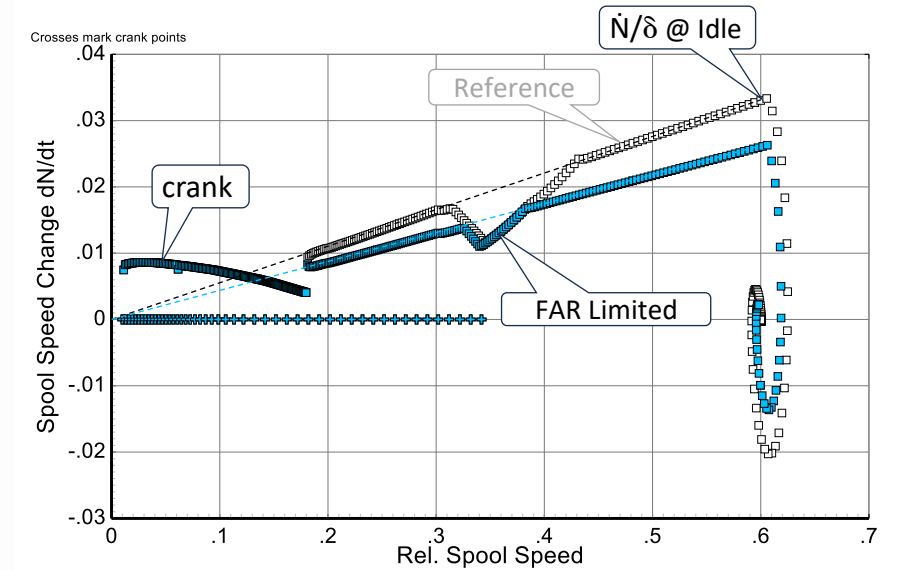
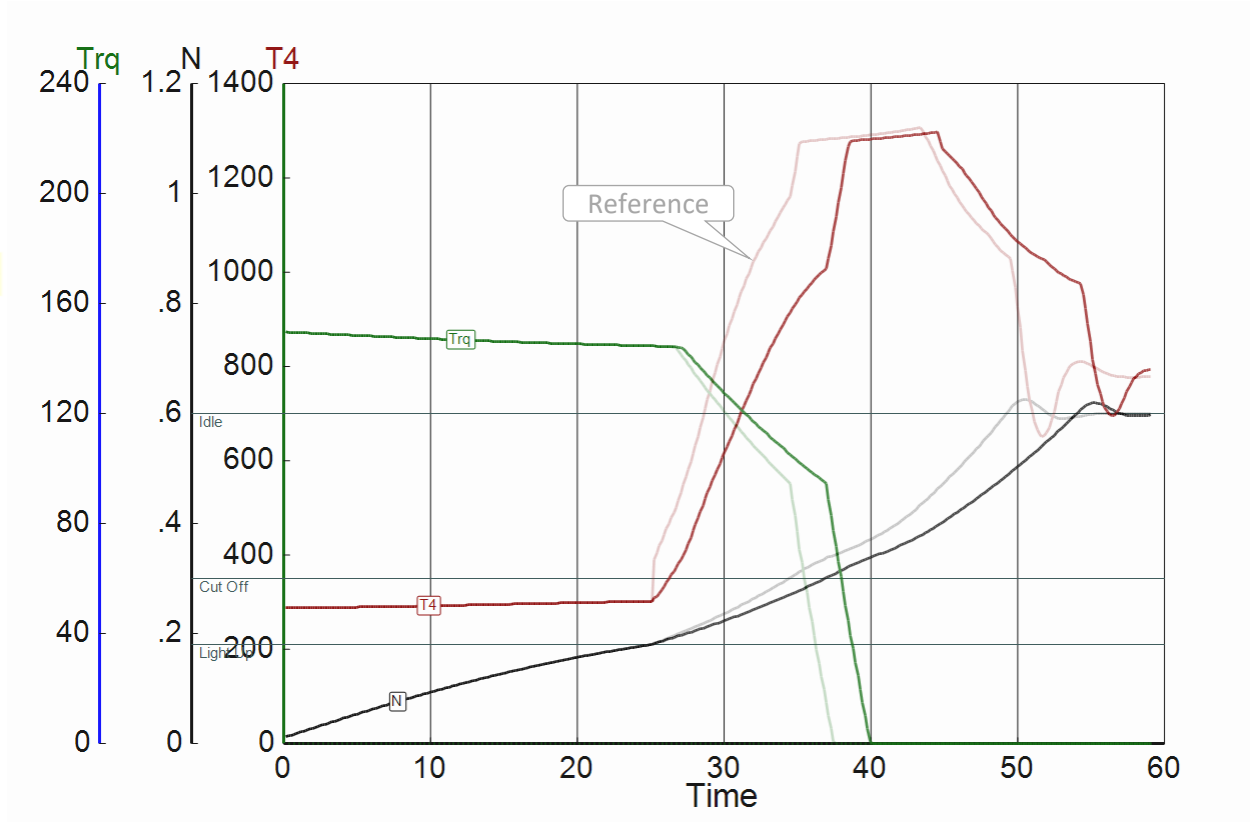
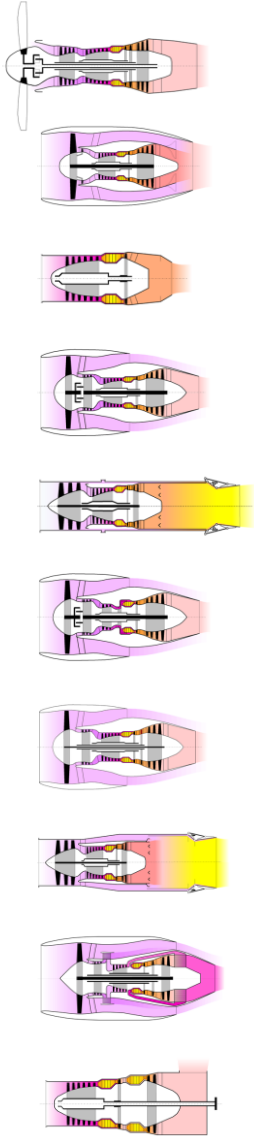
Note:

The scheduled  $\dot{N}/\delta$  increases linearly from light-up to idle speed. This approach prevents an overly large step in  $T_4$  at light-up.

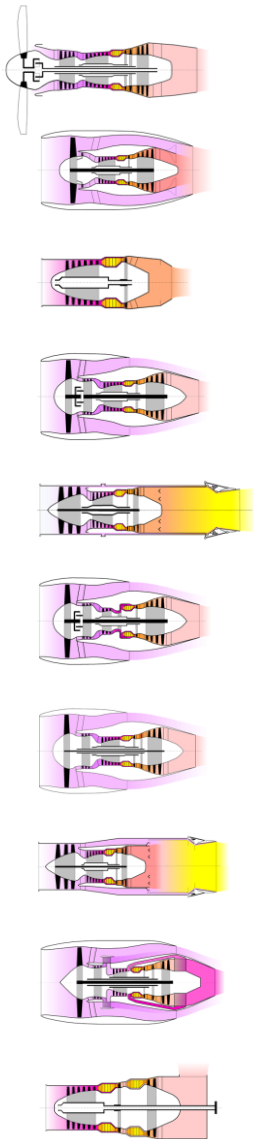


# Acceleration Rate Limiter $\dot{N}/\delta$ @ Idle

## From 0.033 to 0.026

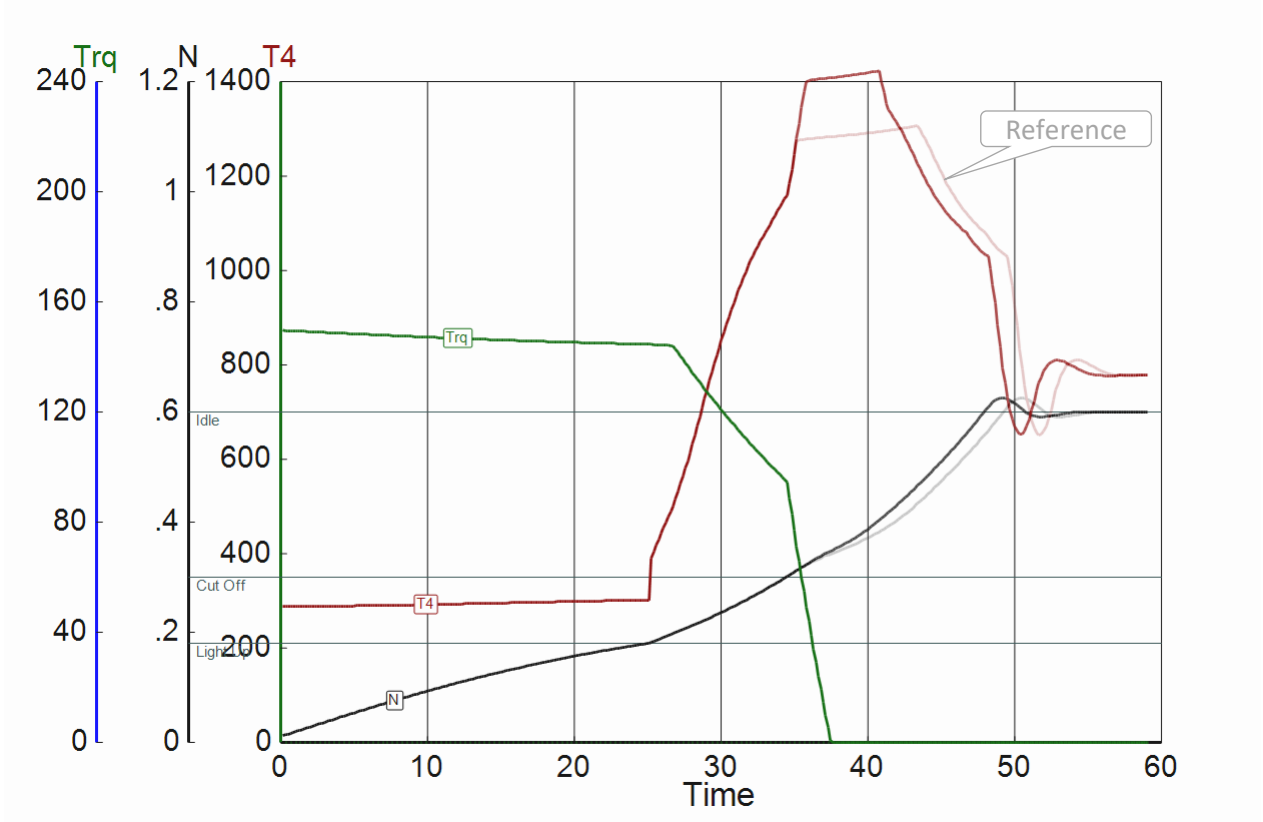


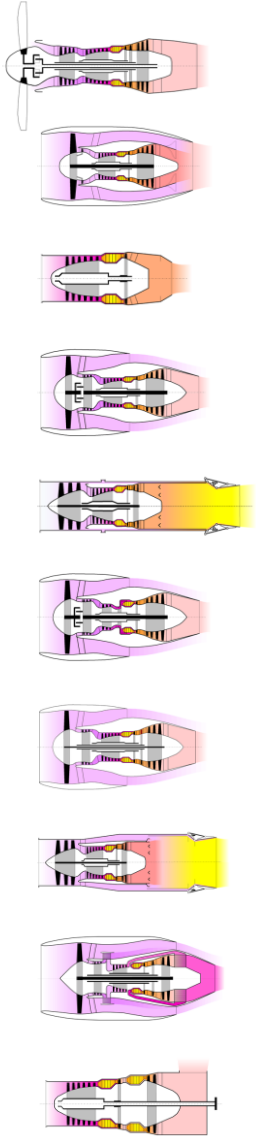




# FAR<sub>max</sub> Limiter

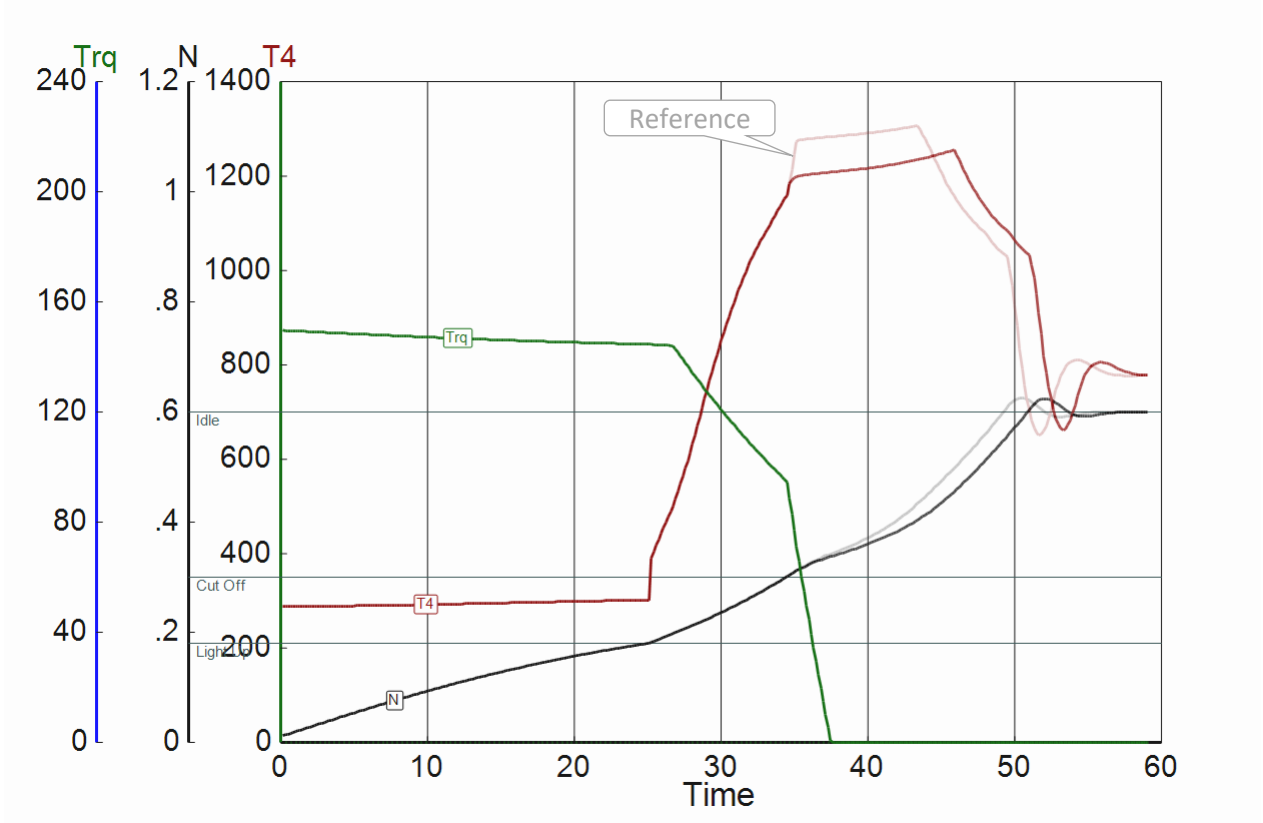
From 0.026 to 0.03



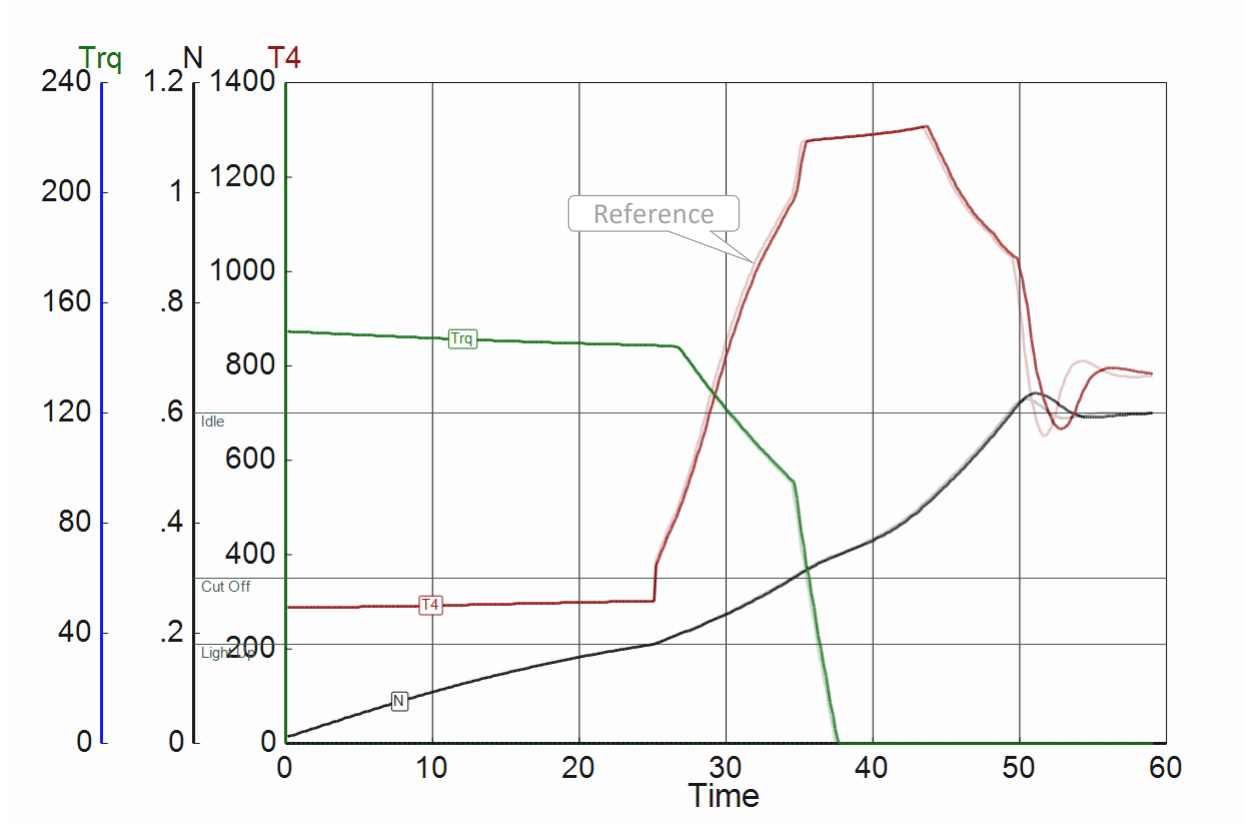
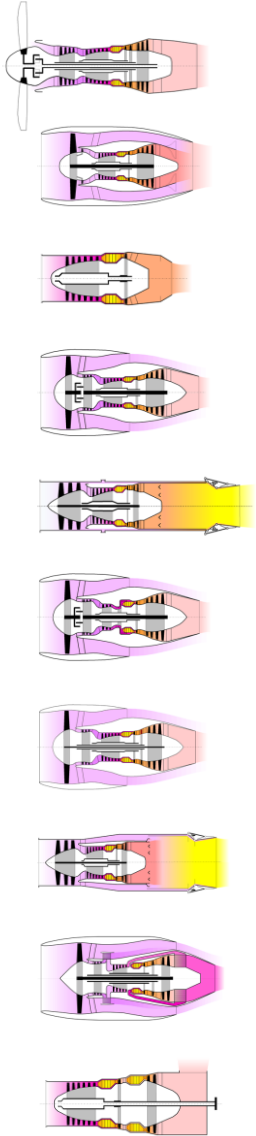


# T<sub>5</sub> Limiter

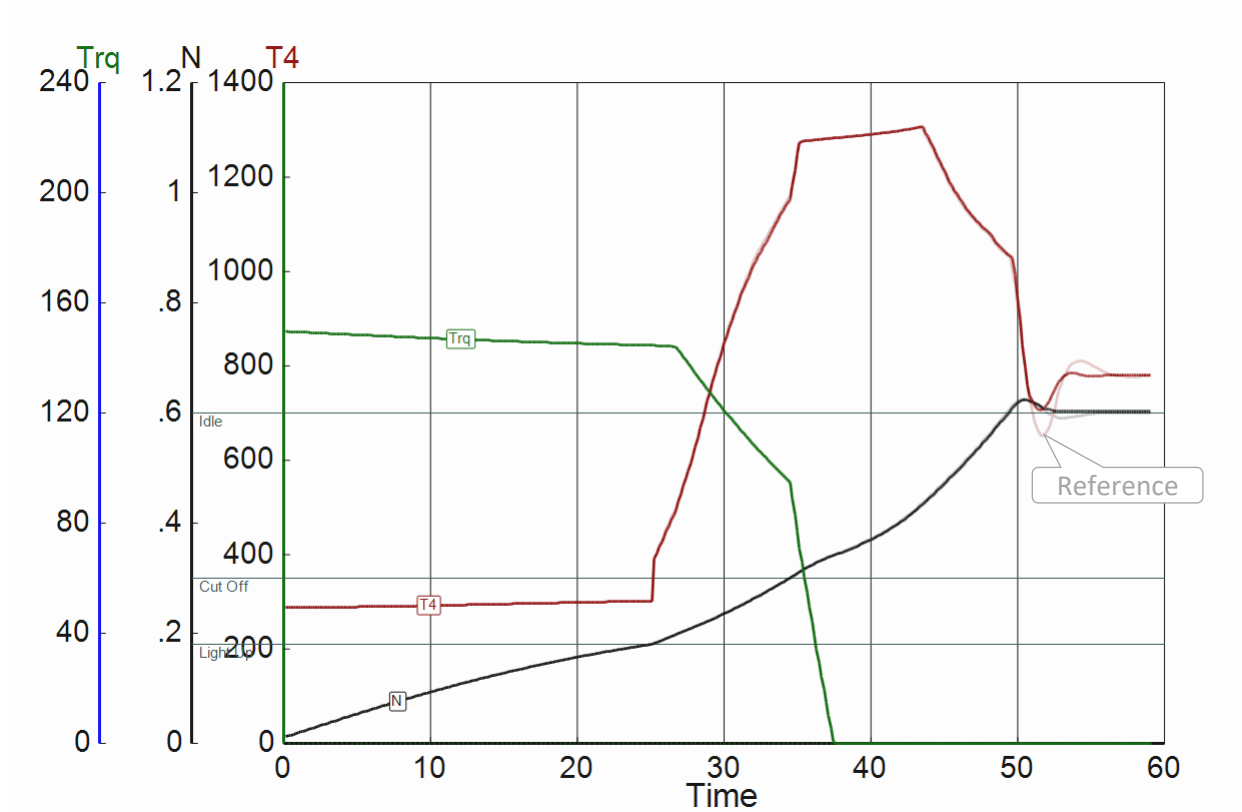
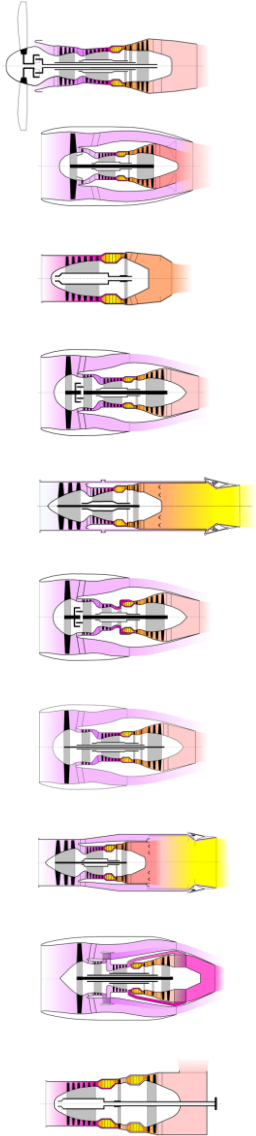
Introduced as 1050K



# Proportional Control Constant From 0.04 to 0.02 (-50%)

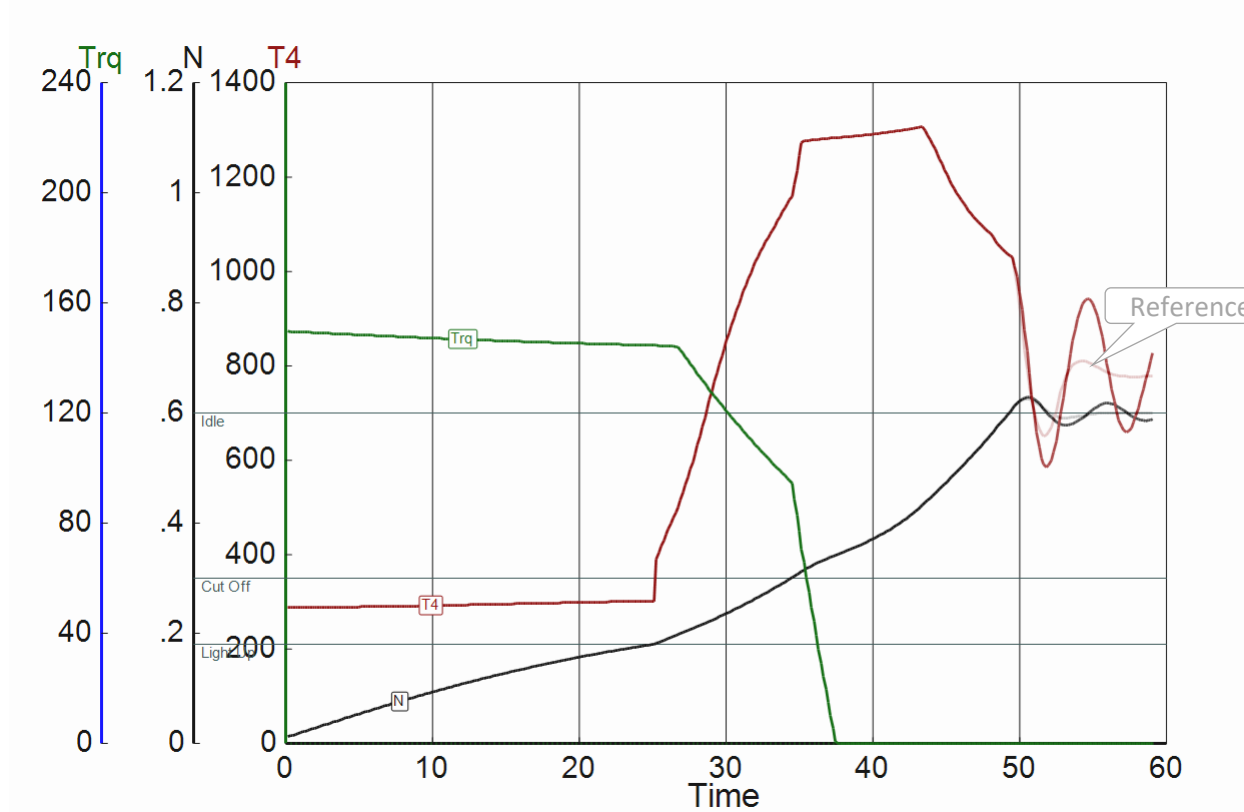
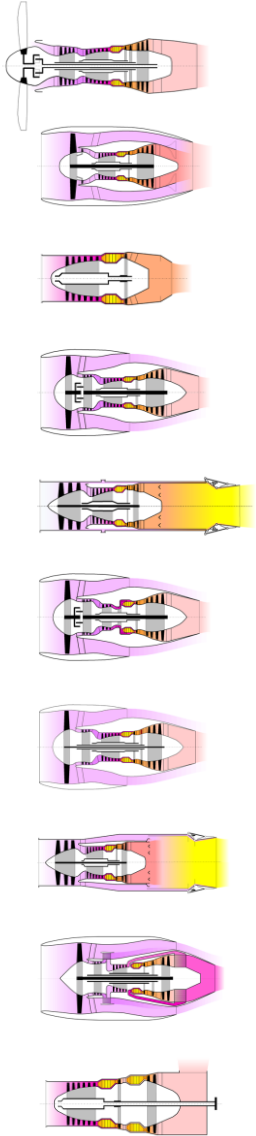


# Integral Control Constant From 0 to 1

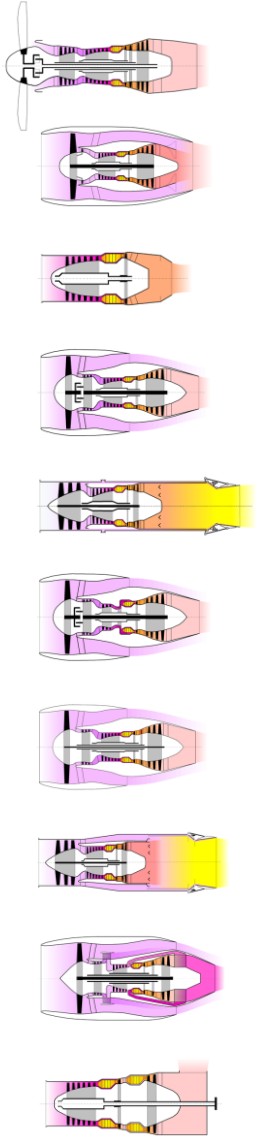


# Differential Control Constant

## From 0.04 to 0

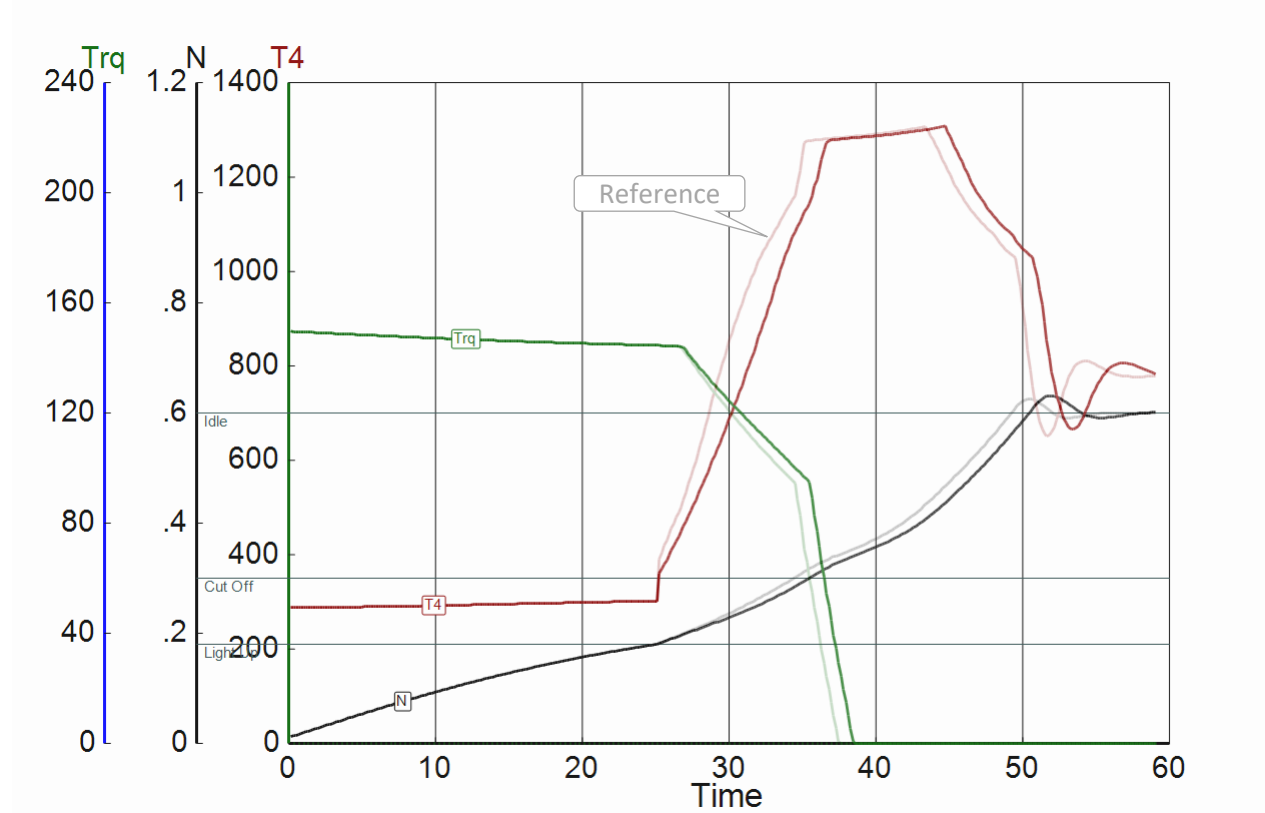






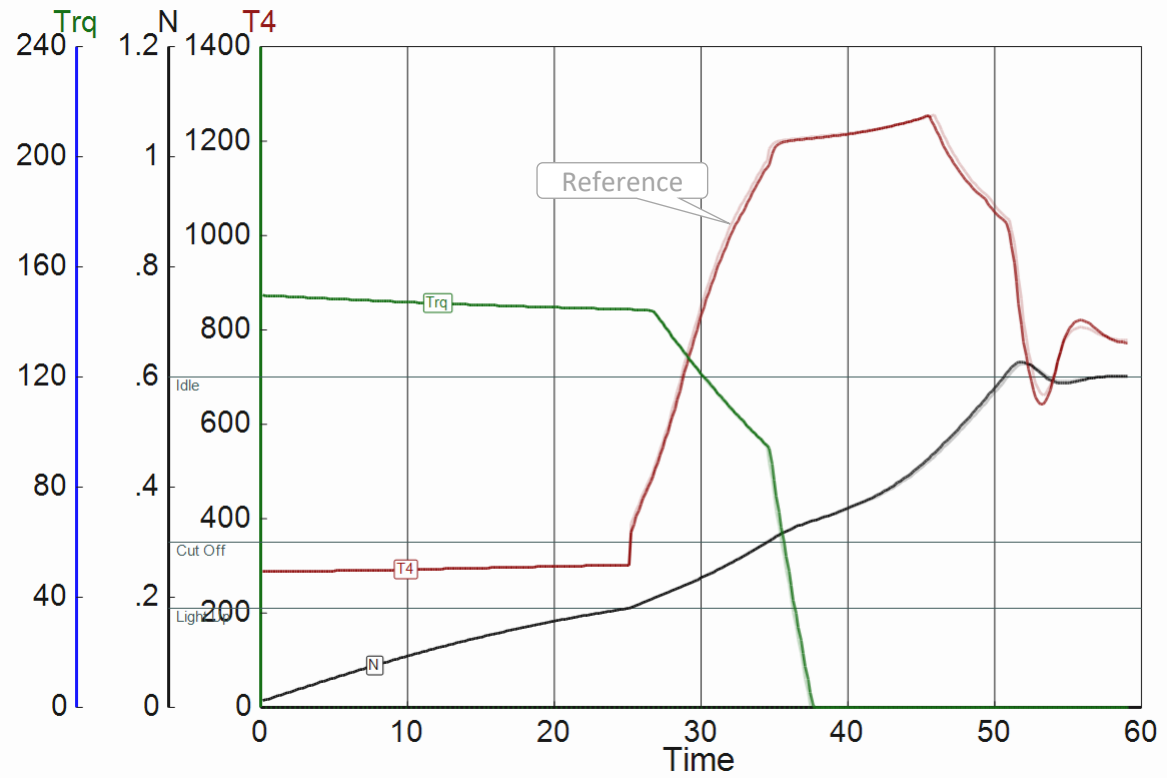
# Gain Modifier

## From 1 to 2



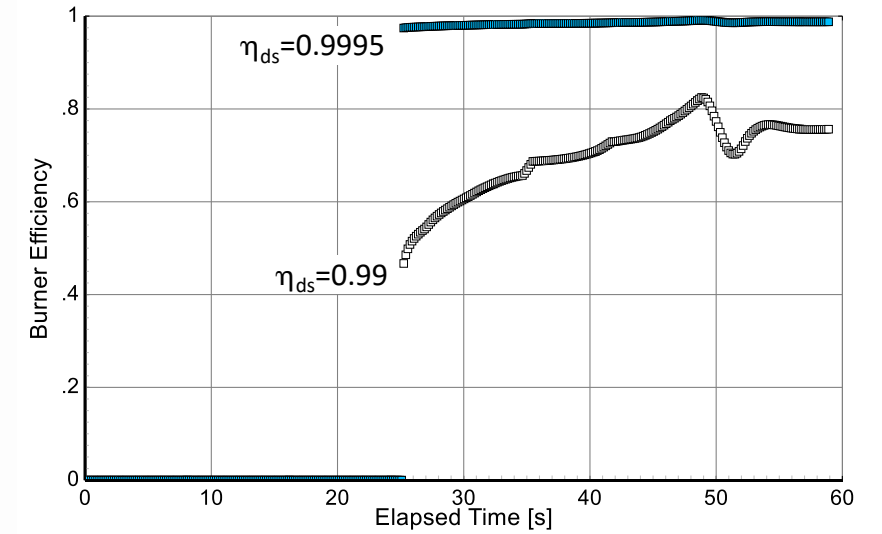
# Effect of Burner Efficiency

## $T_{5,max} = 1050K$



Burner efficiency is calculated as

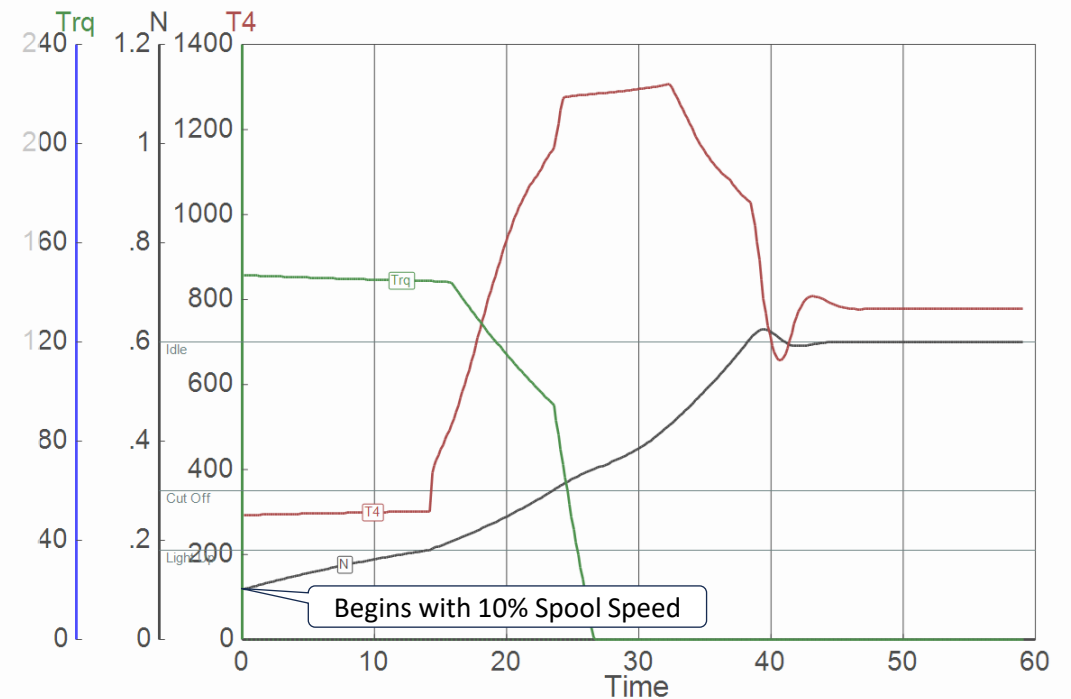
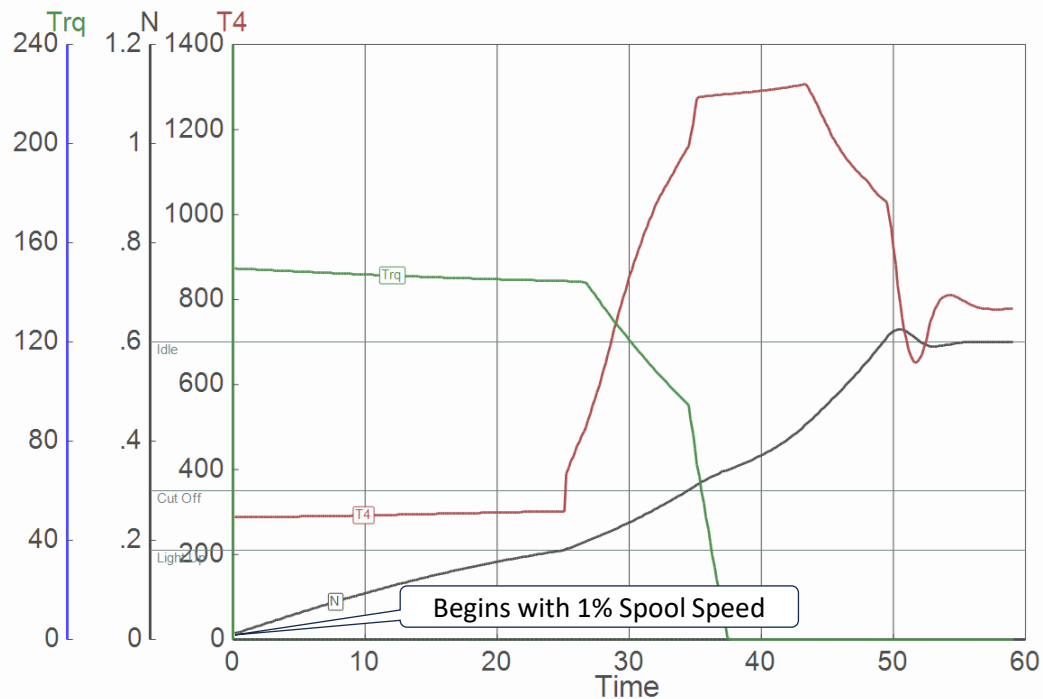
$$\log(1 - \eta) = \log(1 - \eta_{ds}) + 1.6 \times \log(\Omega/\Omega_{ds})$$



# Compressor Map Extension

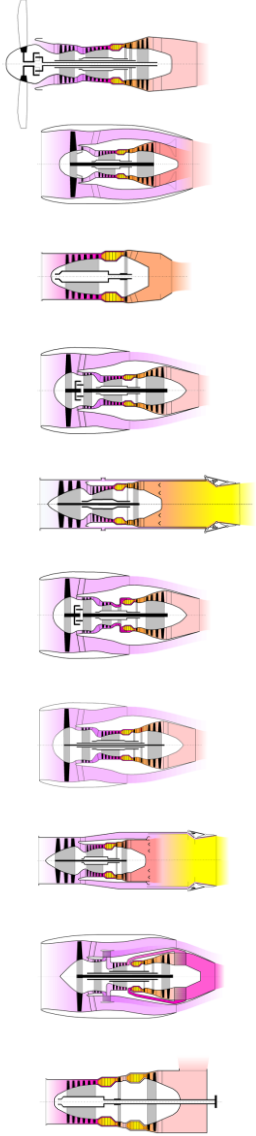
**Reference Map**  
**Split  $\beta$ -Line Grid\***  
 $N/\sqrt{\theta} = 1\% \dots 105\%$

**Conventional Map**  
**Standard  $\beta$ -Line Grid**  
 $N/\sqrt{\theta} = 10\% \dots 105\%$



\* see chapter 18 in "Propulsion and Power", 2<sup>nd</sup> Edition





# Windmilling

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# Preparing a Windmilling Simulation

Create an engine model consisting of

- The thermodynamic cycle reference point
- Extended compressor and turbine maps
  - Downto very low speed (10% ... 5%)
  - Include compressor pressure ratio region with  $P/P < 1$

Find a suitable starting point for the simulation

- A zero fuel flow point at a low spool speed and Mach number  $> 0$  is required
  - Run a zero Mach number steady state case with 30% spool speed
  - Increase Mach number in small steps – fuel flow will decrease.
  - Select a point with low fuel flow
  - Switch to windmilling mode



# Steady State Operation

## 30% Spool Speed

Station	W kg/s	T K	P kPa	WRstd kg/s
amb		288.15	101.325	
1	2.313	288.15	101.325	
2	2.313	288.15	101.325	2.313
3	2.313	337.54	136.838	1.853
31	1.987	337.54	136.838	
4	2.027	1087.70	135.028	2.955
41	2.189	1037.21	135.028	3.116
49	2.189	992.25	102.390	
5	2.327	956.36	102.390	4.196
6	2.327	956.36	102.390	
8	2.327	956.36	102.390	4.196
Bleed	0.025	337.54	136.838	
-----				
P2/P1 = 1.0000	P4/P3 = 0.9868	P6/P5 1.0000		
Efficiencies:	isentropic	polytr	RNI	P/P
Compressor	0.5217	0.5415	1.000	1.350
Burner	0.9826			0.987
Turbine	0.6663	0.6588	0.296	1.319
-----				
Spool mech Eff	0.9900	Speed	30.00 %	
-----				
hum [%]	war0	FHV	Fuel	
0.0	0.00000	42.769	Generic	

SL static, ISA , Rel GG Speed=0.300

FN	=	0.18 kN	Net Thrust
TSFC	=	227.3385 g/(kN*s)	Thrust Specific Fuel Consumption
FN/W2	=	76.22 m/s	Specific Thrust
Prop Eff	=	0.0000	Propulsion Efficiency
eta core	=	0.0039	Core Efficiency
P5/P2	=	1.0105 EPR	Engine Pressure Ratio
WF	=	0.04007 kg/s	Fuel Flow
s NOx	=	0.02662	NOx Severity Parameter
XM8	=	0.1256	Nozzle Throat Mach No.
A8	=	0.0881 m²	Geometric Nozzle Throat Area
P8/Pamb	=	1.0105	Nozzle Pressure Ratio
WBld/w2	=	0.01100	Bleed Air Flow/Mass Flow w2
Ang8	=	16.00 deg	Nozzle Petal Angle
CD8	=	0.9427	Nozzle Discharge Coefficient
WClN/w2	=	0.07000	Turbine Nozzle Guide Vane Cooling Air / w2
WClR/w2	=	0.06000	Turbine Rotor Cooling Air/ w2
Loading	=	919.88 %	Burner Loading in % of the Cycle Design Point Value
e45 th	=	0.65477	Thermodynamic Turbine Efficiency
far8	=	0.01752	Nozzle Throat Fuel-Air-Ratio
PWX	=	0.00 kw	Power Offtake

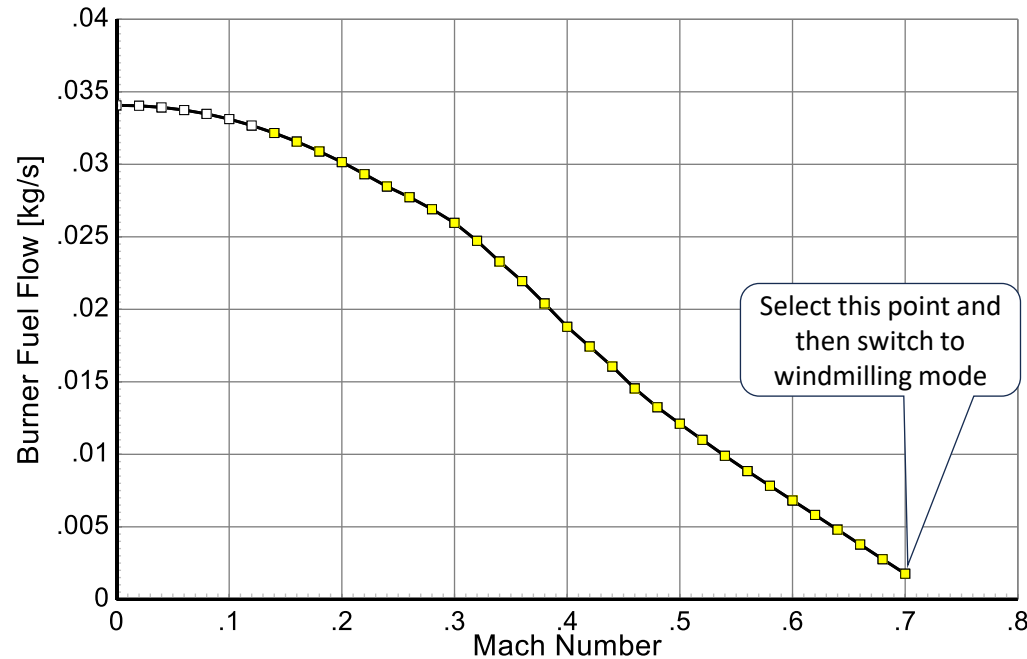
### Standard off-design iteration setup for a given speed

Variable	Error
Beta in HPC Map	Turbine Flow
T4	Turbine Pressure Ratio
Beta in HPT Map	Nozzle Pressure Ratio



# Parametric Study With Increasing Flight Mach Number

Mach Number = 0 ... 0.7



One might encounter convergence problems when switching from a normal, steady-state performance point with a given spool speed to a windmilling point. In such a case, run a parametric study with an increasing Mach number. Select a point with low fuel flow from this study, treat it as a normal off-design point, and then switch to windmilling.



# Steady State Operation

## Windmilling at 30% Spool Speed

Station	W kg/s	T K	P kPa	WRstd kg/s
amb		288.15	101.325	
1	5.859	318.17	143.401	
2	5.859	318.17	143.401	4.350
3	5.859	345.06	171.777	3.782
31	5.033	345.06	171.777	
4	5.033	345.06	162.318	3.438
41	5.443	345.06	162.318	3.718
49	5.443	315.82	103.652	
5	5.795	317.59	103.652	5.947
6	5.795	317.59	103.652	
8	5.795	317.59	103.652	5.947
Bleed	0.064	345.06	171.779	
-----				
P2/P1 = 1.0000	P4/P3 = 0.9449	P6/P5 1.0000		
Efficiencies:	isent	polytr	RNI	P/P
Compressor	0.6242	0.6336	1.258	1.198
Burner	0.0000			0.945
Turbine	0.7067	0.6933	1.293	1.566
-----				
Spool mech Eff	0.9900	Speed	30.00 %	
-----				
hum [%]	war0	FHV	Fuel	
0.0	0.00000	42.769	Generic	

SL Mn=0.722 ISA , Rel GG Speed=0.300, **windmilling**

FN	=	-1.07 kN	Net Thrust
TSFC	=	0.0000 g/(kN*s)	Thrust Specific Fuel Consumption
FN/w2	=	-182.24 m/s	Specific Thrust
Prop Eff	=	1.5855	Propulsion Efficiency
eta core	=	0.0000	Core Efficiency
P5/P2	=	0.7228 EPR	Engine Pressure Ratio
<b>WF</b>	<b>=</b>	<b>0.00000 kg/s</b>	<b>Fuel Flow</b>
s NOx	=	0.00000	NOx Severity Parameter
XM8	=	0.1805	Nozzle Throat Mach No.
A8	=	0.0881 m²	Geometric Nozzle Throat Area
P8/Pamb	=	1.0230	Nozzle Pressure Ratio
WBld/w2	=	0.01100	Bleed Air Flow/Mass Flow w2
Ang8	=	16.00 deg	Nozzle Petal Angle
CD8	=	0.9154	Nozzle Discharge Coefficient
WC1N/w2	=	0.07000	Turbine Nozzle Guide Vane Cooling Air / w2
WC1R/w2	=	0.06000	Turbine Rotor Cooling Air/ w2
Loading	=	1509.42 %	Burner Loading in % of the Cycle Design Point value
e45 th	=	0.00000	Thermodynamic Turbine Efficiency
far8	=	0.00000	Nozzle Throat Fuel-Air-Ratio
PWX	=	0.00 kw	Power Offtake

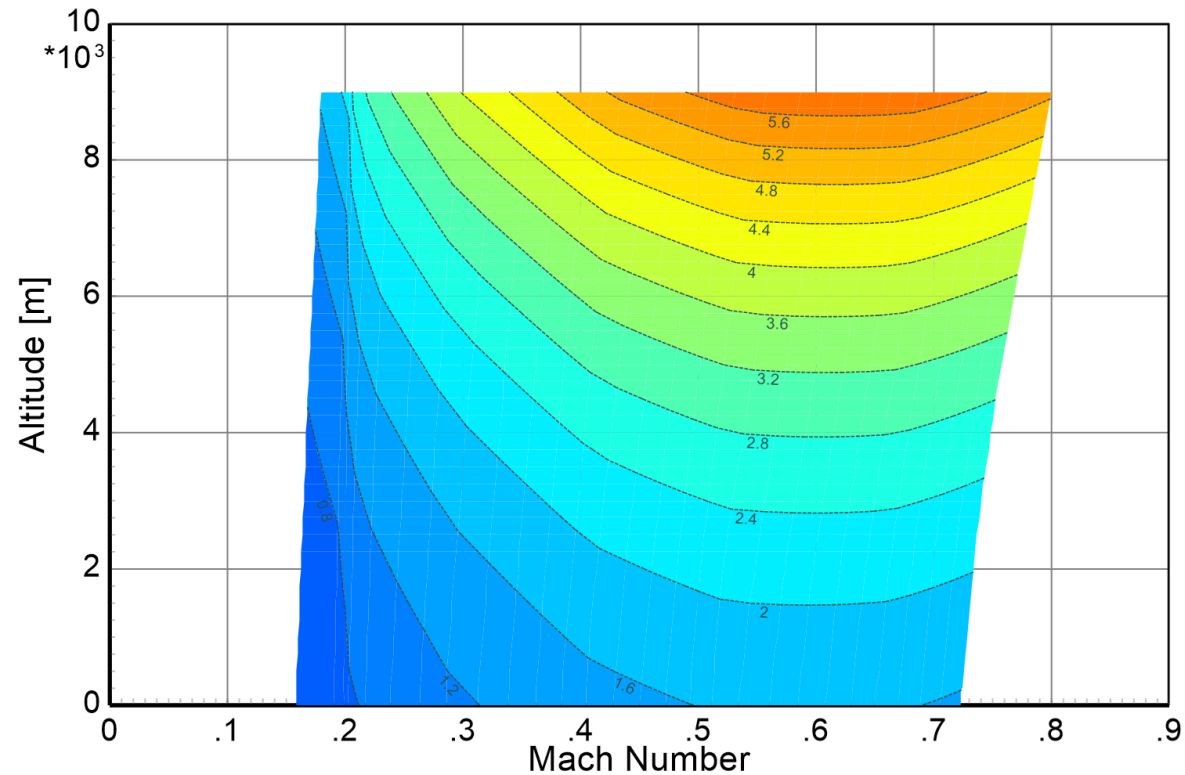
### Windmilling iteration setup, spool speed is given

Variable	Error
Beta in HPC Map	Turbine Flow
Mach number XM	Turbine Pressure Ratio
Beta in HPT Map	Nozzle Pressure Ratio



# Windmilling

## Burner Loading Relative to Idle Sea Level Static



Burner loading during windmilling is an indicator of the difficulty encountered during a relight, which can be a significant challenge.

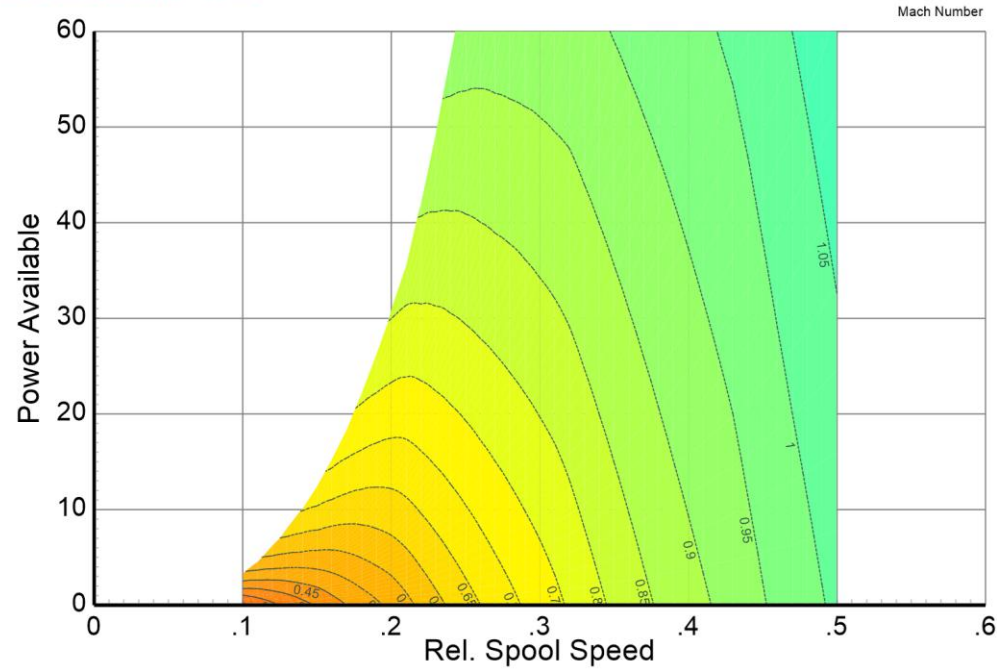


# Windmilling

## Shaft Power Available = $f(N, \text{Mach})$

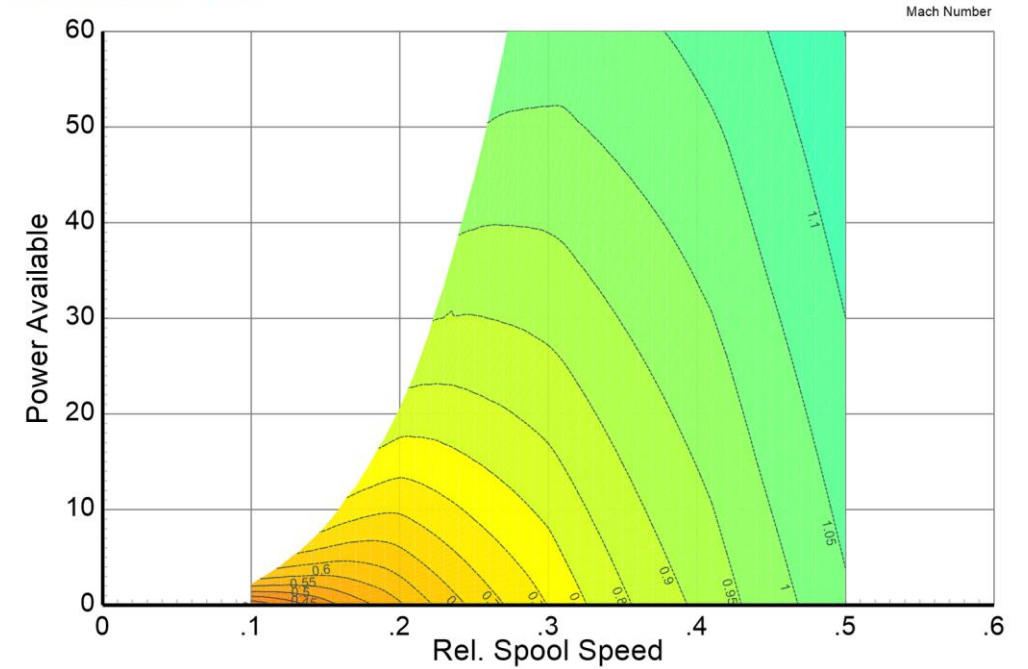
Altitude 1000m, ISA

HPC Spool Speed Z<sub>XN</sub> = 0.5 ... 0.1  
Mechanical Efficiency = 1 ... 0.6



Altitude 5000m, ISA

HPC Spool Speed Z<sub>XN</sub> = 0.5 ... 0.1  
Mechanical Efficiency = 1 ... 0.6



Note:

Power Available is calculated as  $(1 - \eta_{\text{mech}}) * P_{W_{\text{Turbine}}}$

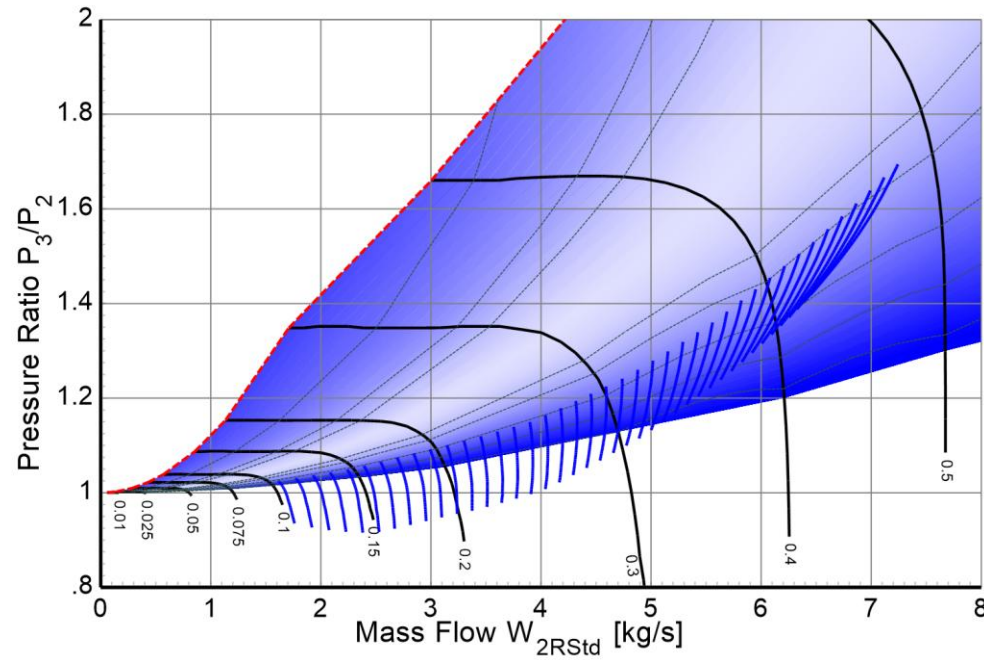




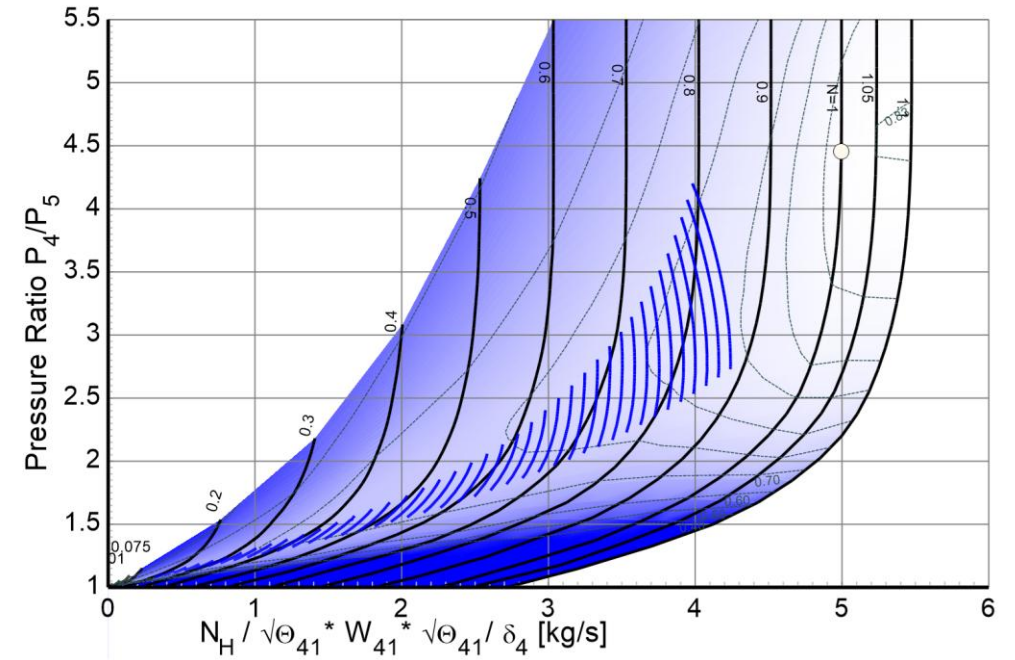
# Windmilling in the Component Maps

$$PW_X = (1 - \eta_{\text{mech}}) * PW_{\text{Turbine}}$$

HPC Spool Speed Z<sub>XN</sub> = 0.5 ... 0.1  
Mechanical Efficiency = 1 ... 0.6

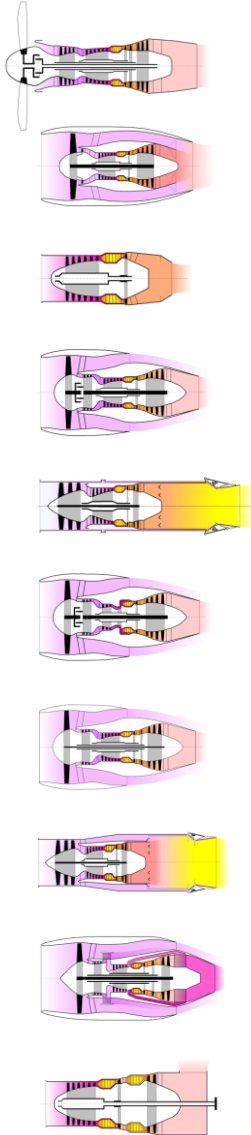


HPC Spool Speed Z<sub>XN</sub> = 0.5 ... 0.1  
Mechanical Efficiency = 1 ... 0.6



# A Windmilling Relight Example

## Altitude 6000m, Mach=0.44



Station	W kg/s	T K	P kPa	WRstd kg/s
amb		249.15	47.181	
1	1.354	258.80	53.881	
2	1.354	258.80	53.881	2.413
3	1.354	266.07	56.903	2.317
31	1.163	266.07	56.903	
4	1.163	266.07	55.727	2.032
41	1.258	266.07	55.727	2.198
49	1.258	258.16	47.377	
5	1.339	258.64	47.377	2.714
6	1.339	258.64	47.377	
8	1.339	258.64	47.377	2.714
Bleed	0.015	266.07	56.903	
-----				
P2/P1 = 1.0000	P4/P3 = 0.9793	P6/P5 1.0000		
Efficiencies:	isent	polytr	RNI	P/P
Compressor	0.5596	0.5630	0.604	1.056
Burner	0.0000			0.979
Turbine	0.6552	0.6499	0.604	1.176
-----				
Spool mech Eff	0.9900	Speed	15.00 %	
-----				
hum [%]	war0	FHV	Fuel	
0.0	0.00000	42.769	Generic	

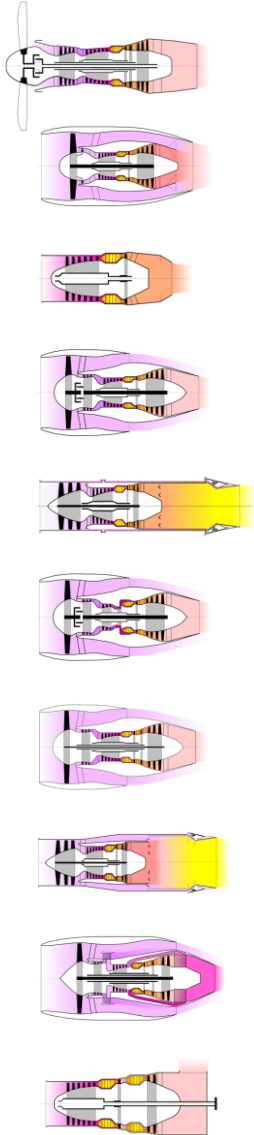
Alt= 6000m / Mn=0.440 ISA , Rel GG Speed=0.150, **windmilling**

FN	=	-0.16 kN	Net Thrust
TSFC	=	0.0000 g/(kN*s)	Thrust Specific Fuel Consumption
FN/W2	=	-114.61 m/s	Specific Thrust
Prop Eff	=	1.6975	Propulsion Efficiency
eta core	=	0.0000	Core Efficiency
P5/P2	=	0.8793 EPR	Engine Pressure Ratio
WF	=	0.00000 kg/s	<b>Fuel Flow</b>
s NOx	=	0.00000	NOx Severity Parameter
XM8	=	0.0769	Nozzle Throat Mach No.
A8	=	0.0881 m²	Geometric Nozzle Throat Area
P8/Pamb	=	1.0042	Nozzle Pressure Ratio
WBld/W2	=	0.01100	Bleed Air Flow/Mass Flow W2
Ang8	=	16.00 deg	Nozzle Petal Angle
CD8	=	0.9639	Nozzle Discharge Coefficient
WClN/W2	=	0.07000	Turbine Nozzle Guide Vane Cooling Air / W2
WClR/W2	=	0.06000	Turbine Rotor Cooling Air/ W2
Loading	=	1684.95 %	Burner Loading in % of the Cycle Design Point Value
e45 th	=	0.00000	Thermodynamic Turbine Efficiency
far8	=	0.00000	Nozzle Throat Fuel-Air-Ratio
PWX	=	0.00 kW	Power offtake

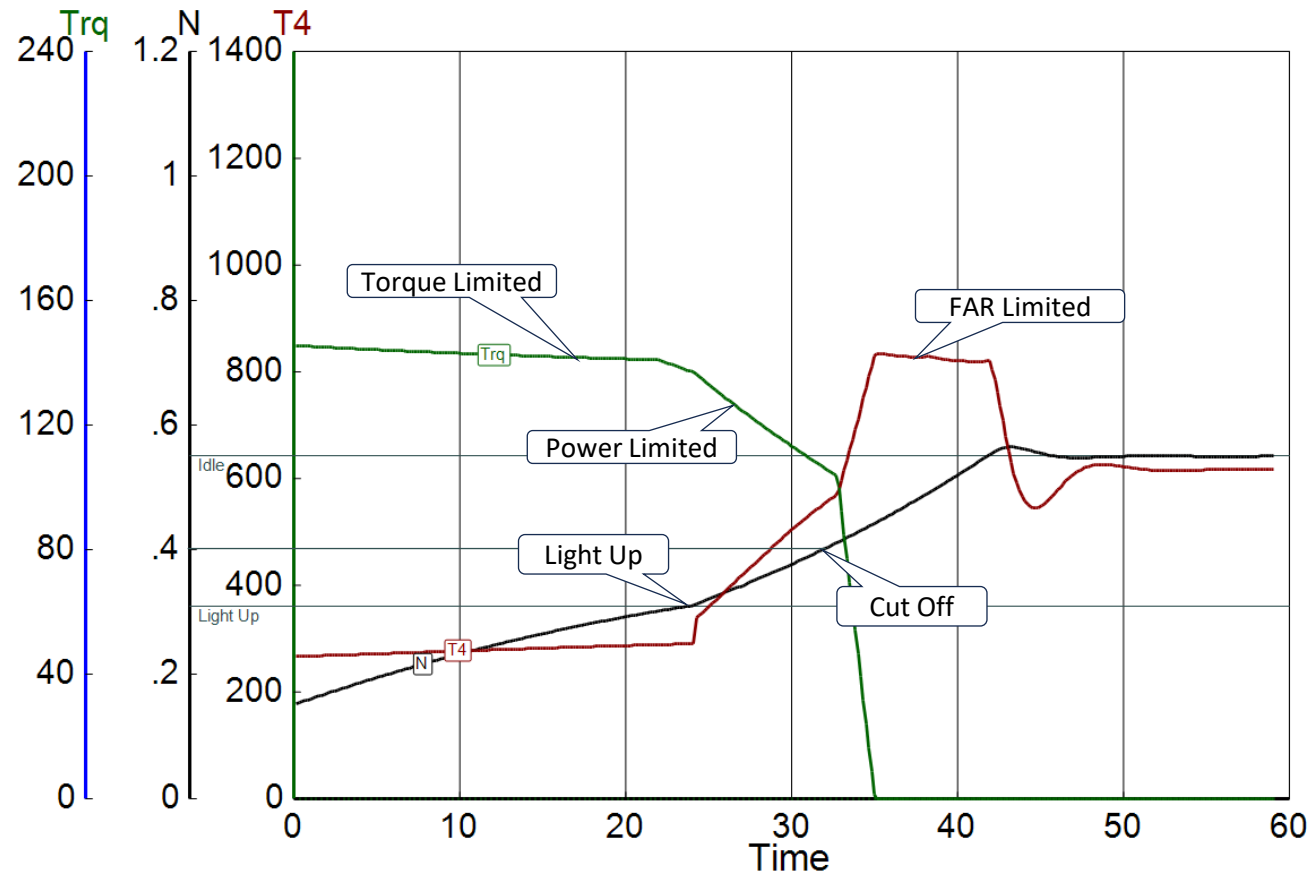
### Windmilling iteration setup, spool speed is given

Variable	Error
Beta in HPC Map	Turbine Flow
Mach number XM	Turbine Pressure Ratio
Beta in HPT Map	Nozzle Pressure Ratio





# Windmill Relight

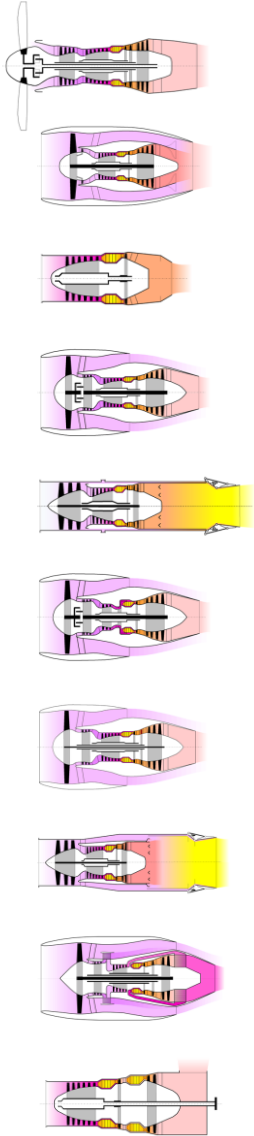


Adjust the numbers for light-up, starter cut-off speed and max starter power:

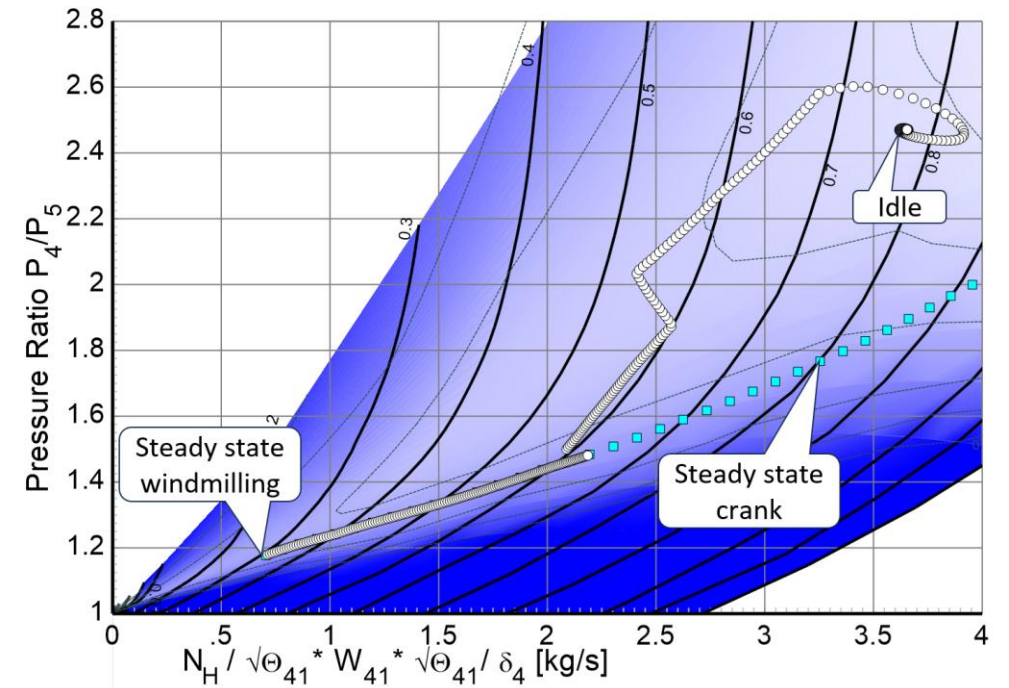
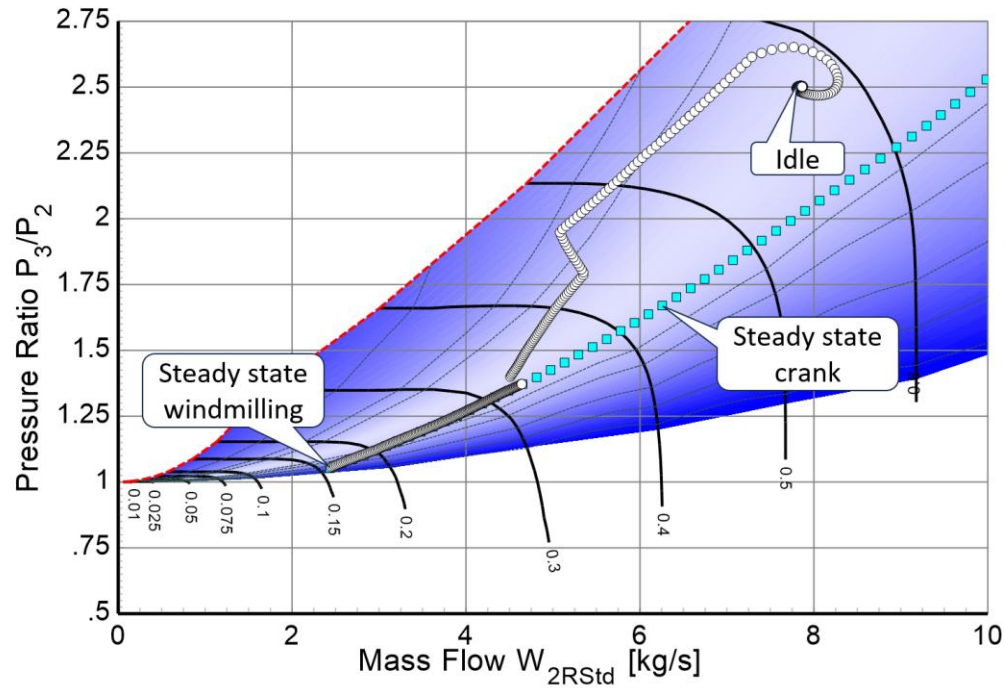
Rotor Inertia	kg m <sup>2</sup>	5.69
Rel N for PLA 0% (Idle)		0.6
Proportional Control Constant		0.04
Integral Control Constant		0
Differential Control Constant		0.04
Gain Modifier		1
Minimum Fuel-Air-Ratio		0.003
Maximum Fuel-Air-Ratio		0.026
Decel (dN/dt)/(P2/Pstd) Limit		0.1
Accel (dN/dt)/(P2/Pstd) Limit		0.033
Rel. Burner Light-up Speed		<b>0.31</b>
Rel. Starter Cut-off Speed		<b>0.4</b>
Max Starter Torque	N m	150
Starter Torque Slope		-0.2
Max Starter Power	kW	<b>60</b>

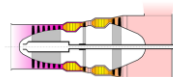
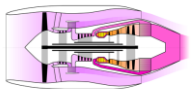
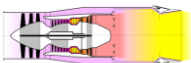
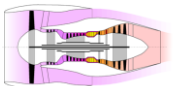
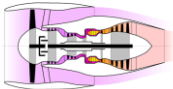
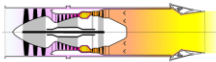
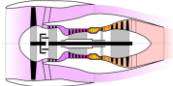
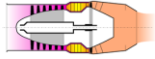
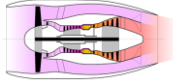
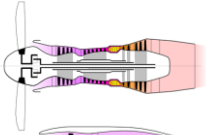






# Windmill Relight





## Concluding Remarks

This tutorial focused on the most basic type of gas turbine: a single-spool machine.

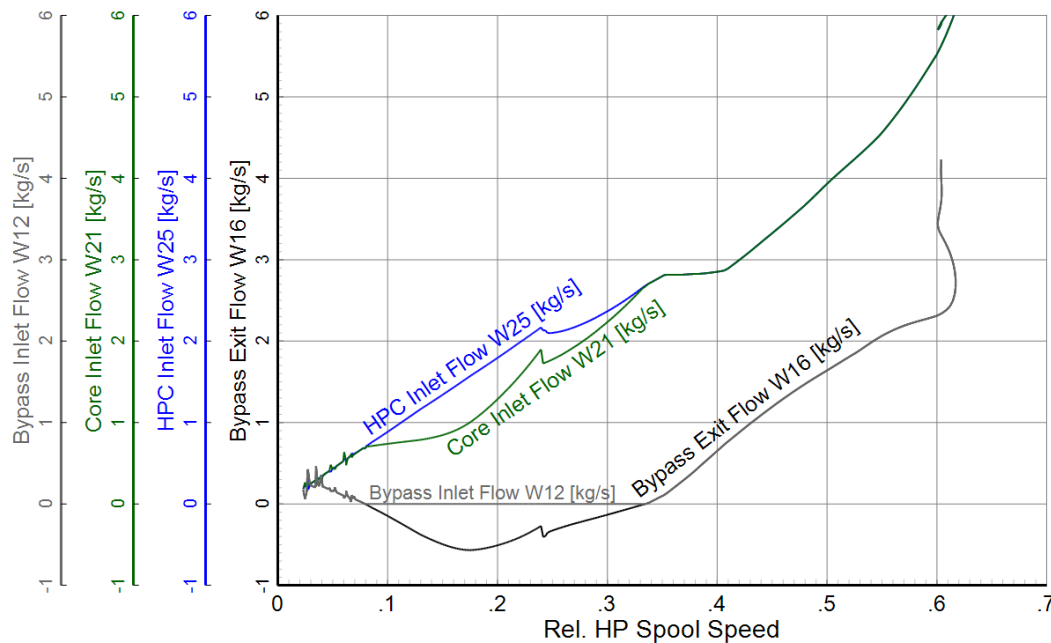
- The starting process for more complex engines with multiple spools is similar when the starter motor is connected to the gas generator spool. The compressors and turbines on the other spools are accelerated by the gas generator's exhaust.
- Windmilling of high bypass ratio turbofans is dominated by the behavior of the fan.
- However, the procedures for simulating starting and windmilling remain the same.



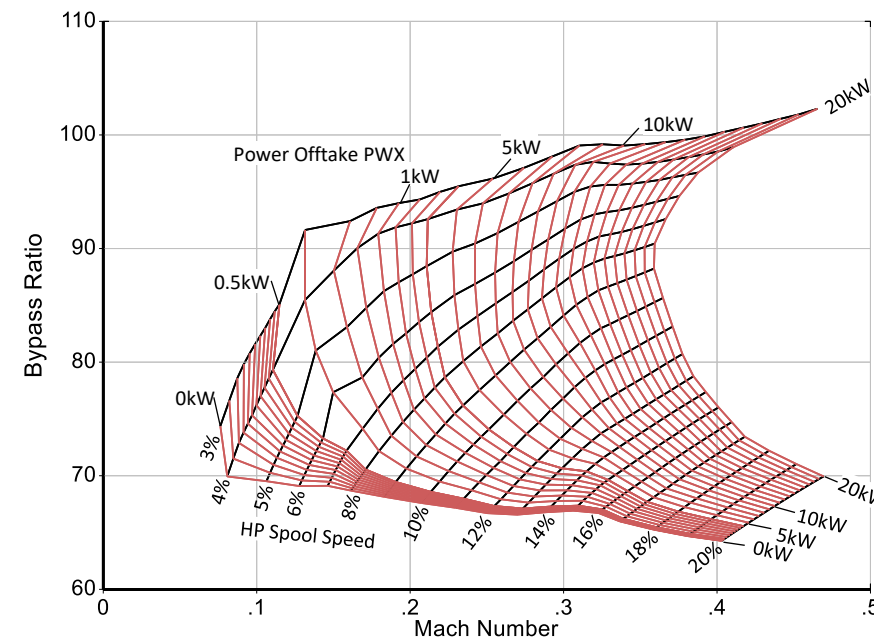


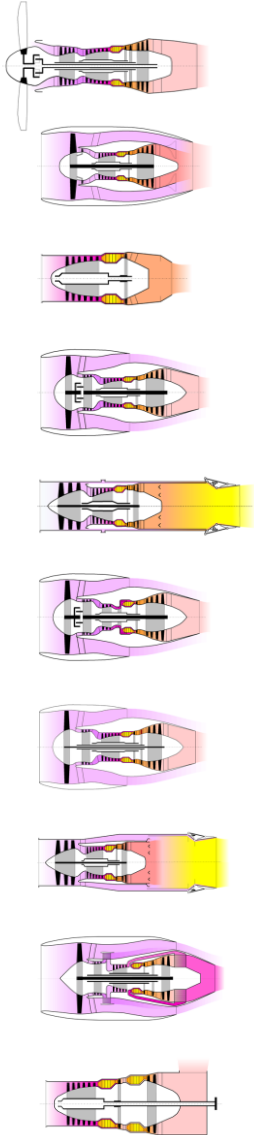
# Two Interesting Starting and Windmilling Simulation Examples

During the start of a low-bypass ratio mixed-flow turbofan, it's possible that the high pressure compressor (HPC) will draw for some time more flow than the fan can deliver because its spool speed lags behind.



Windmilling of a high bypass ratio turbofan with power offtake from the gas generator.





# Reference

**SPRINGER NATURE**



Joachim Kurzke, Ian Halliwell, Robert Hill

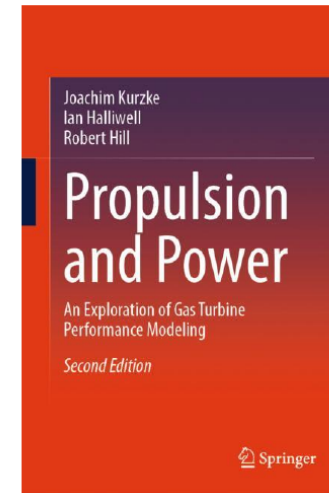
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